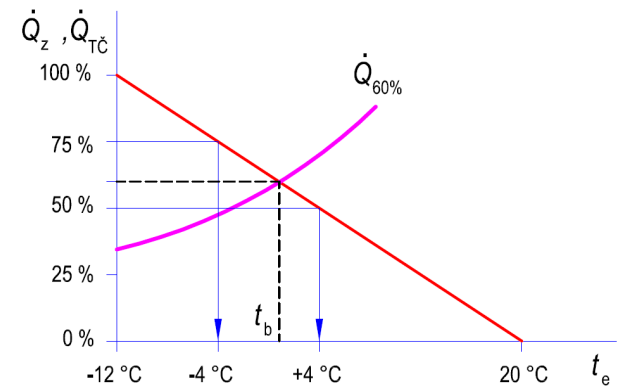




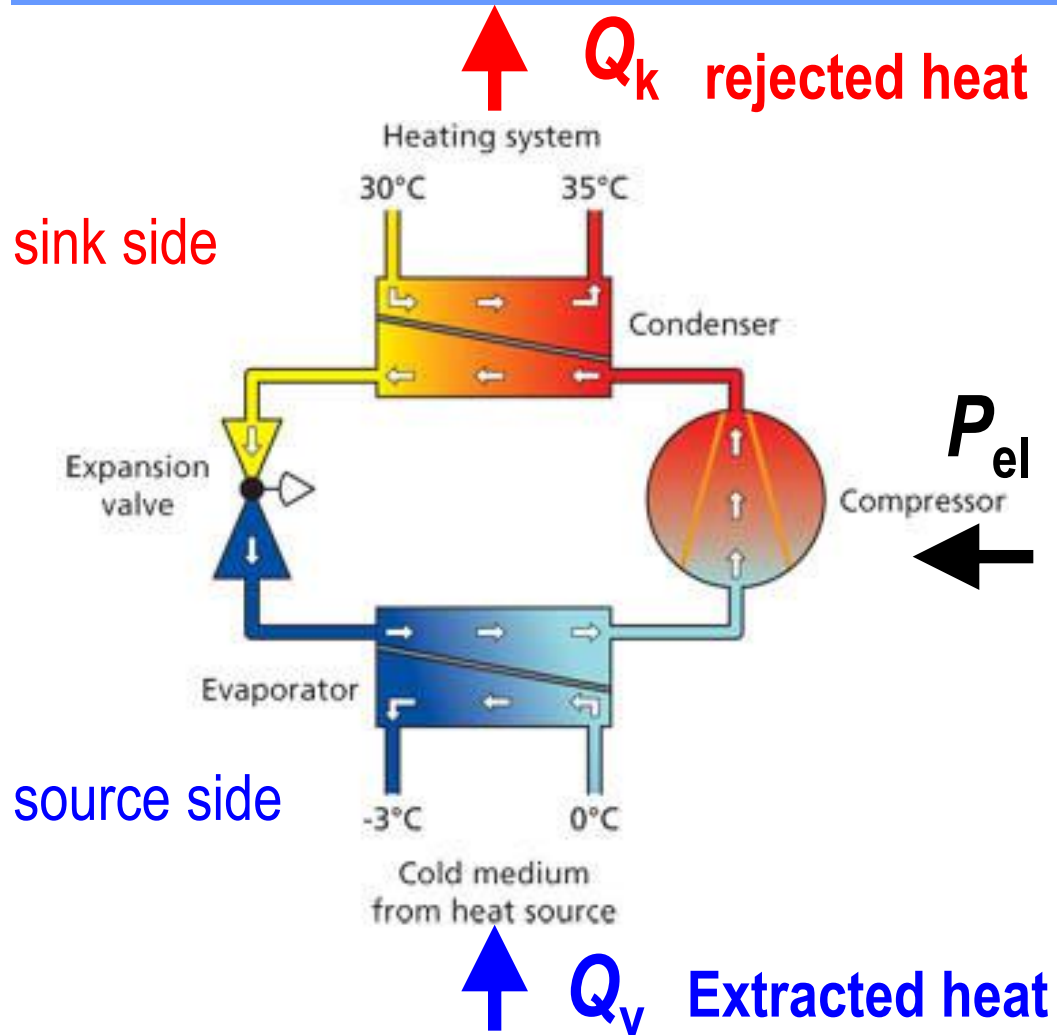
# 7 Heat pumps sizing

- heat pump characteristics
- testing & operation conditions
- balance point
- storage size
- hydraulics





# Heat pump



$$\dot{Q}_k = P_{el} + \dot{Q}_v$$

$$COP = \frac{\dot{Q}_k}{P_{el}}$$

$$\dot{Q}_v = \dot{Q}_k - P_{el}$$

$$\dot{Q}_v = \dot{Q}_k \cdot \left( 1 - \frac{1}{COP} \right)$$



# Heat pumps: air source (ambient)





# Heat pumps: ground source (borehole)





# Heat pumps: ground source (ground HX)





# Heat pumps: water source (water well)

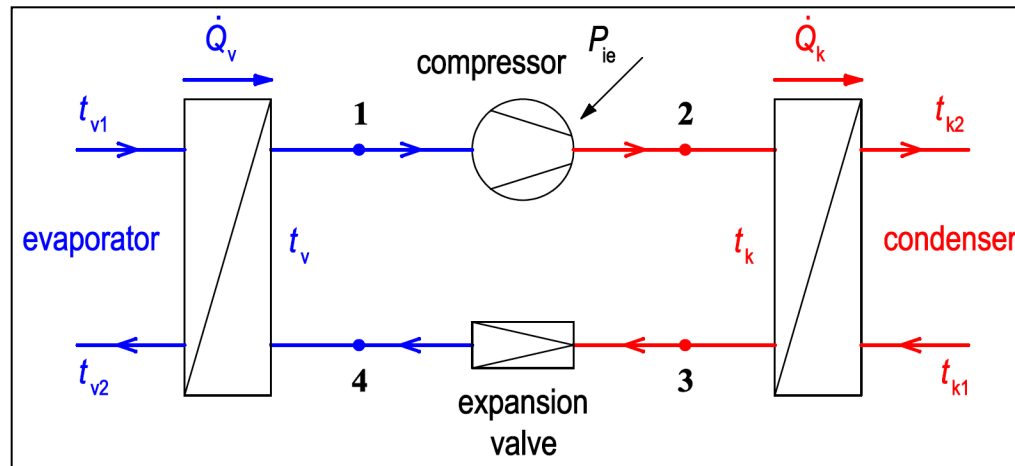




# Heat pump parameters

electric power

$P_{el}$  [kW]



heat output, heat capacity

$Q_k$  [kW]

= heat output from **condenser**

coefficient of performance COP [-]

evaporator  
input  
 $Q_v$  [kW]  
= source output

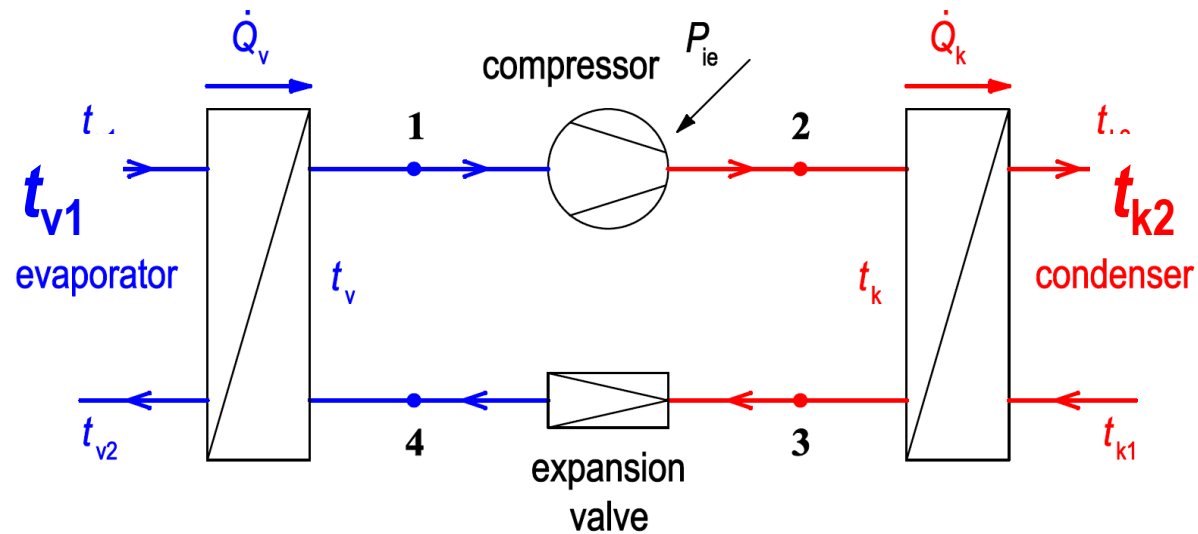


# Heat pump parameters

at given boundary conditions

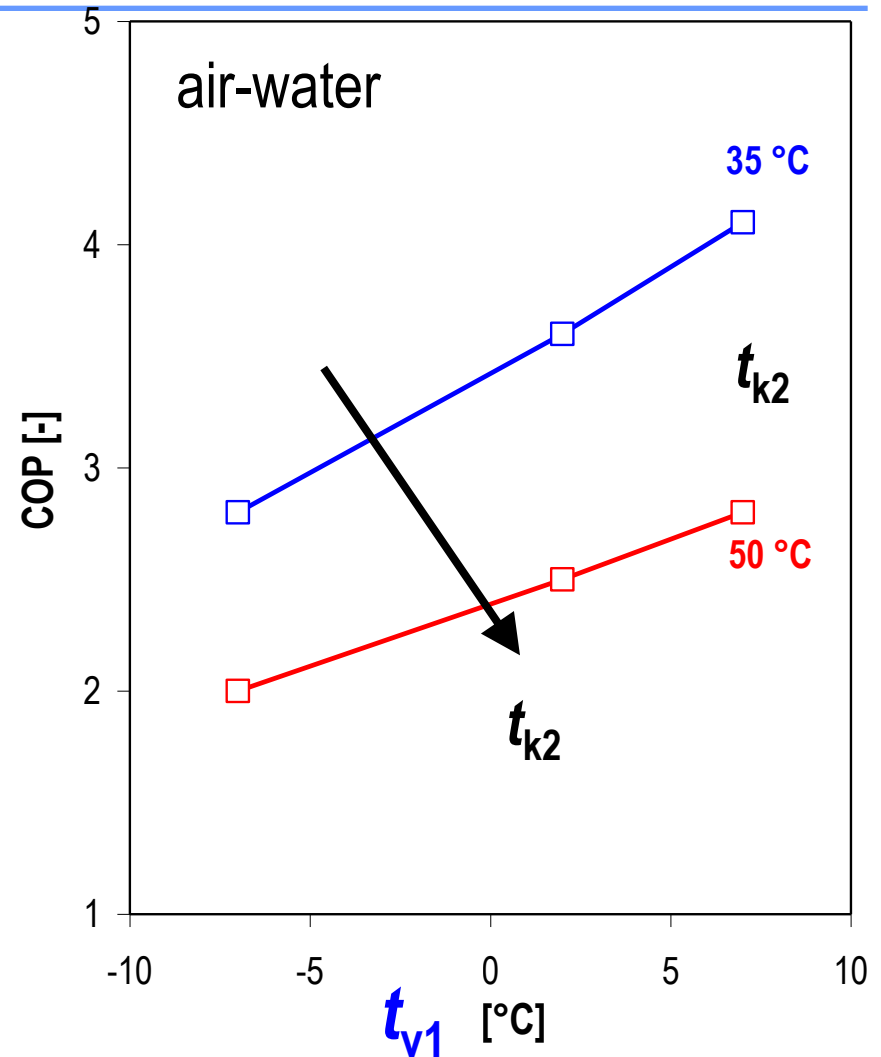
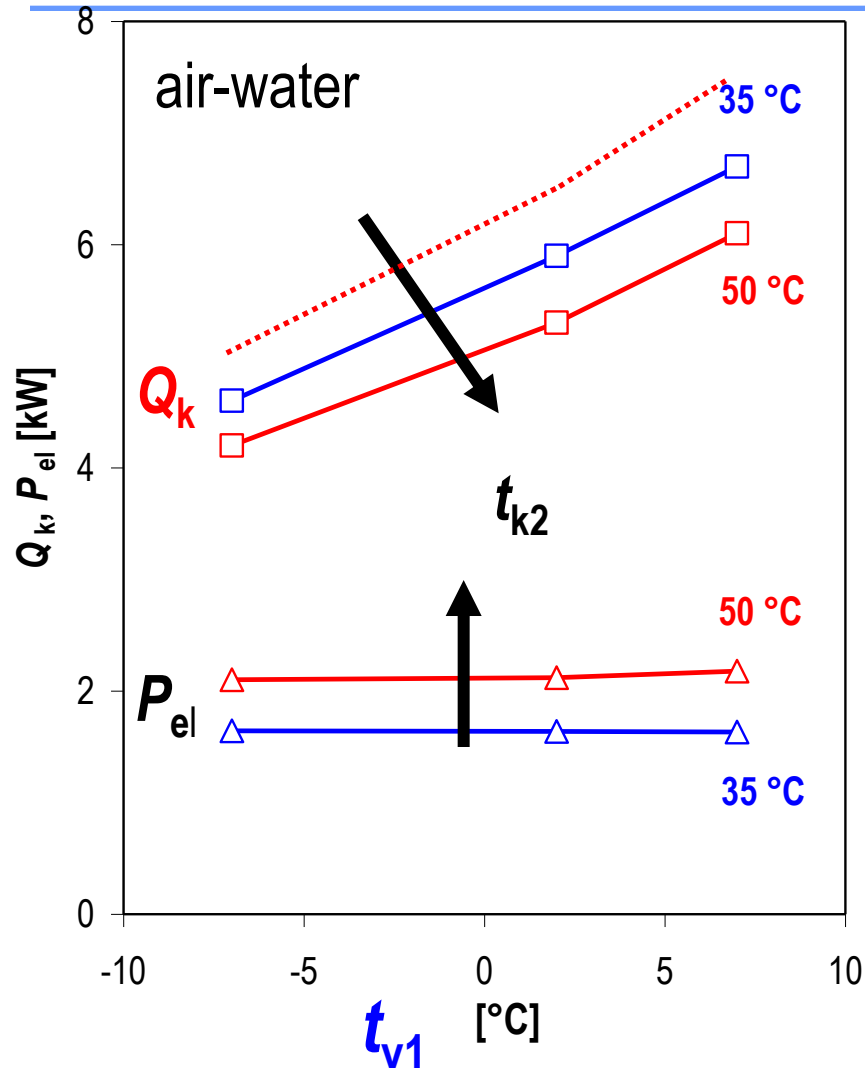
$t_{v1}$  - evaporator input temperature

$t_{k2}$  - condenser output temperature





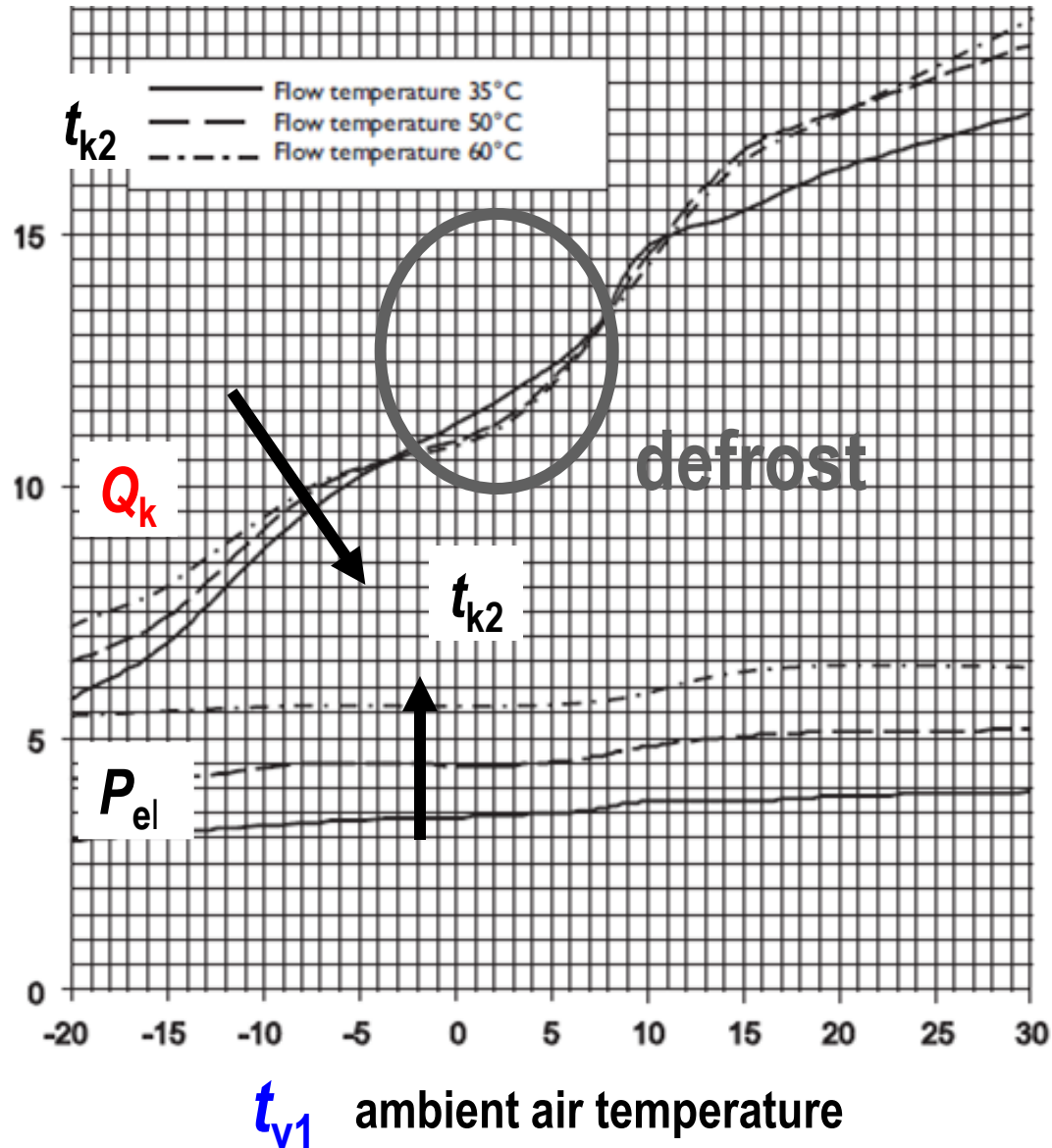
# Air-water heat pump characteristics





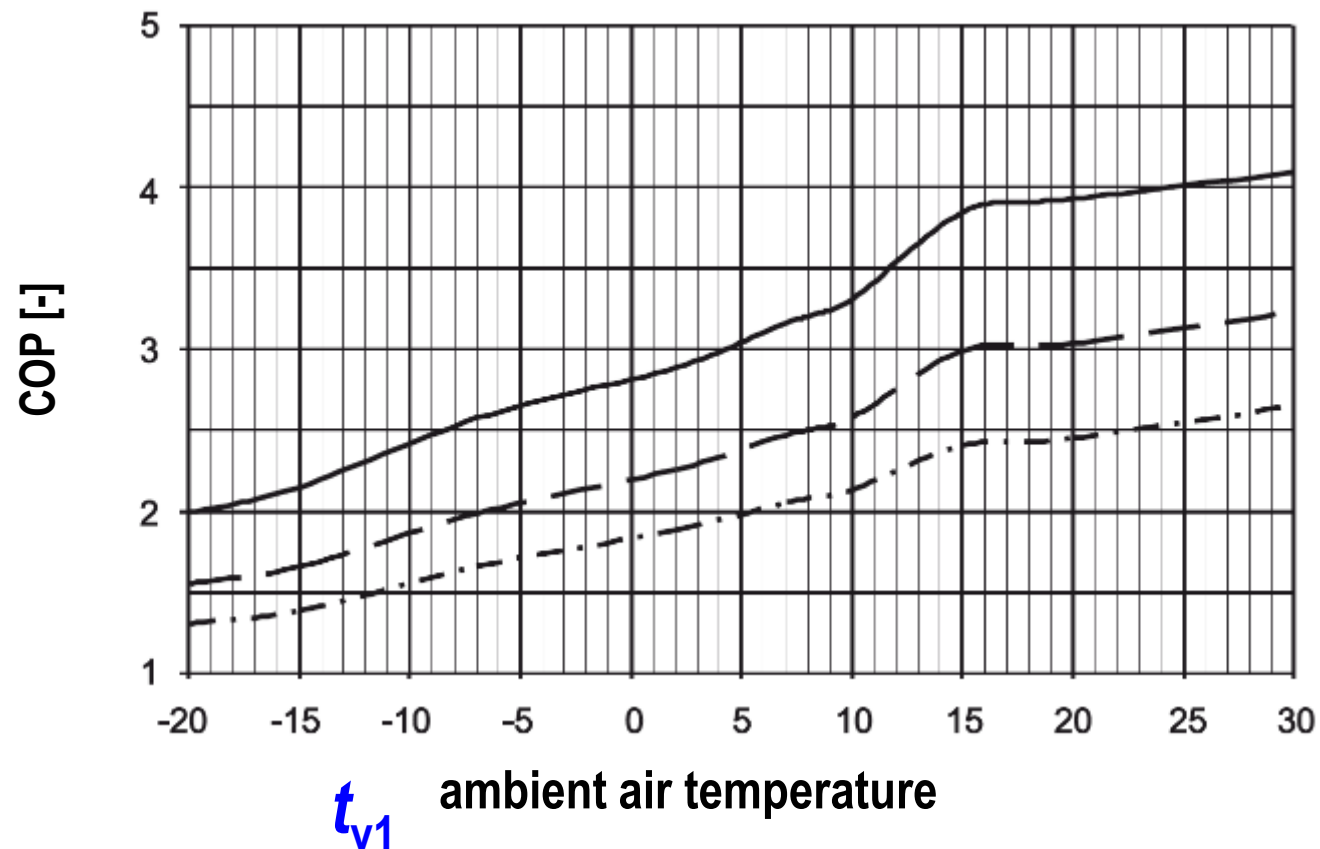
# Air-water heat pump characteristics

electric power, heating capacity [kW]



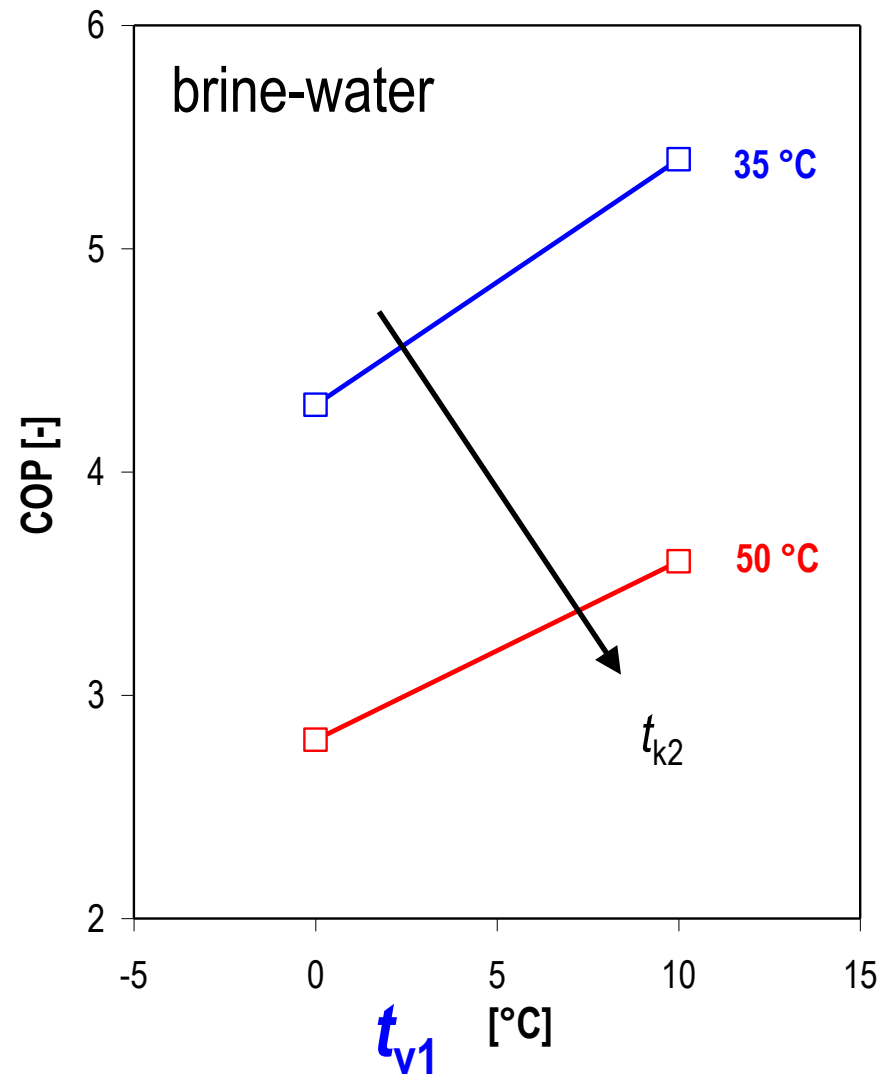
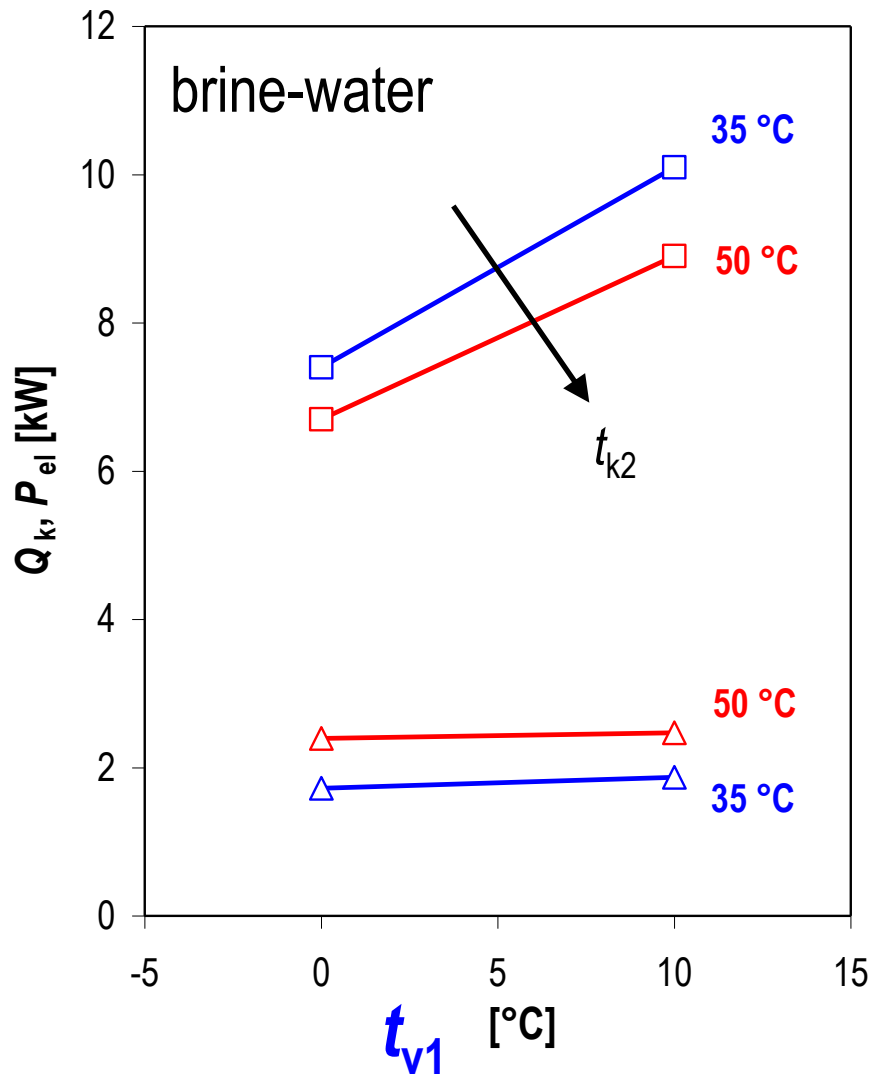


# Air-water heat pump characteristics



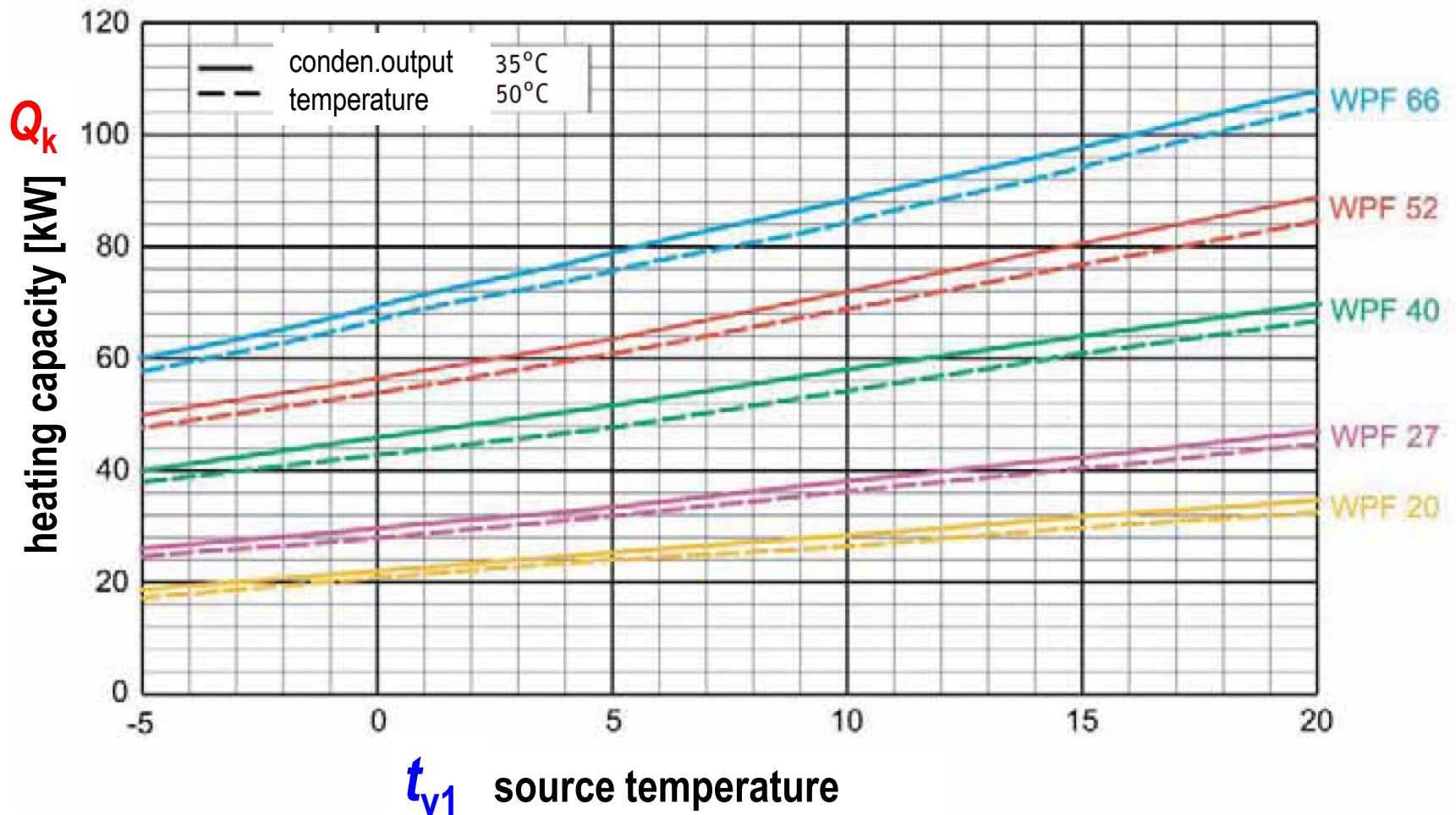


# Brine-water heat pump (ground source)



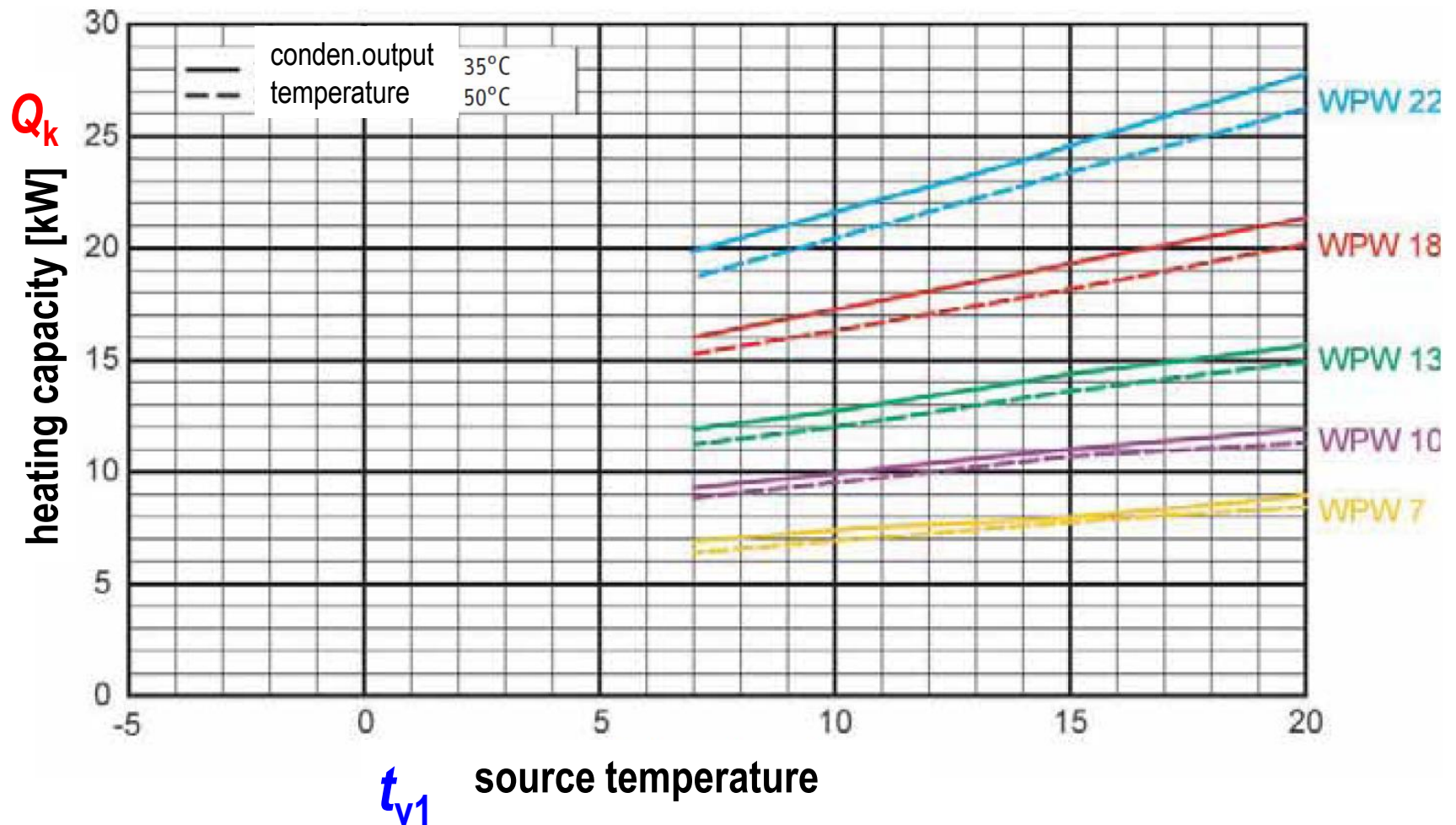


# Brine-water heat pump (ground source)





# Water-water heat pump





# Testing of heat pumps

---

- **EN 14511** - Air conditioners, liquid chilling packages and heat pumps with electrically driven compressors for space heating and cooling.
  - EN 14511-1: dtto - Terms and definitions
  - EN 14511-2: dtto - Test conditions
  - EN 14511-3: dtto – Test methods
  - EN 14511-4: dtto - Requirements
  
- **EN 16147** – Heat pumps with electrically driven compressors. Testing and requirements for marking of domestic hot water units



# Testing conditions

- EN 14511: water-water **W/W** (W10 / W35)
  - **nominal:** 10/35 °C 10/45 °C
  - operation: 15/45 °C 10/55 °C
- EN 14511: brine-water (ground-water) **B/W** (B0 / W35)
  - **nominal:** 0/35 °C 0/45 °C
  - operation: 5/35 °C 5/45 °C 0/55 °C  
-5/45 °C
- EN 14511: air-water (ambient air) **A/W** (A2 / W35)
  - **nominal:** 7/35 °C 7/45 °C
  - operation: 2/35 °C 2/45 °C 7/55 °C  
-7/35 °C -7/45 °C -7/55 °C  
-15/35 °C -15/45 °C



# Requirements on heat pump

- **Quality label EHPA (European Heat Pump Association)**
- minimum *COP* from testing according to EN 14511 in respected lab

**brine-water**                      **B0/W35**                      ***COP* > 4.3**

**water-water**                      **W10/W35**                      ***COP* > 5.1**

**air-water**                      **A2/W35**                      ***COP* > 3.1**



- declaration of sound power level
- documentation: planning, service and operation guides in local language
- customer service network, 24 h reaction time on customer complaints
- 2 years full warranty, spare parts inventory available for 10 years in stock



# Requirements on heat pump – other norms

---

- **EN 14825(SCOP)**
  - **Air to air** - Needs to fulfill min. energy efficiency class A according to EU Regulations, i.e. SCOP shall not be lower than 3.4.
- **EN 15879-1:**
  - **Direct Exchange ground** coupled to Water E4/W35 - 4.30
- **EN 16147:**
  - **Domestic Hot Water** heat pump





# Requirements on heat pump – KEYMARK

---

- Example ... for low temperature heat pumps:

A7/W35;

A20/W35;

B0/W35;

W10/W35 according to EN 14511

bivalent temperature conditions according to EN 14825 for average climate



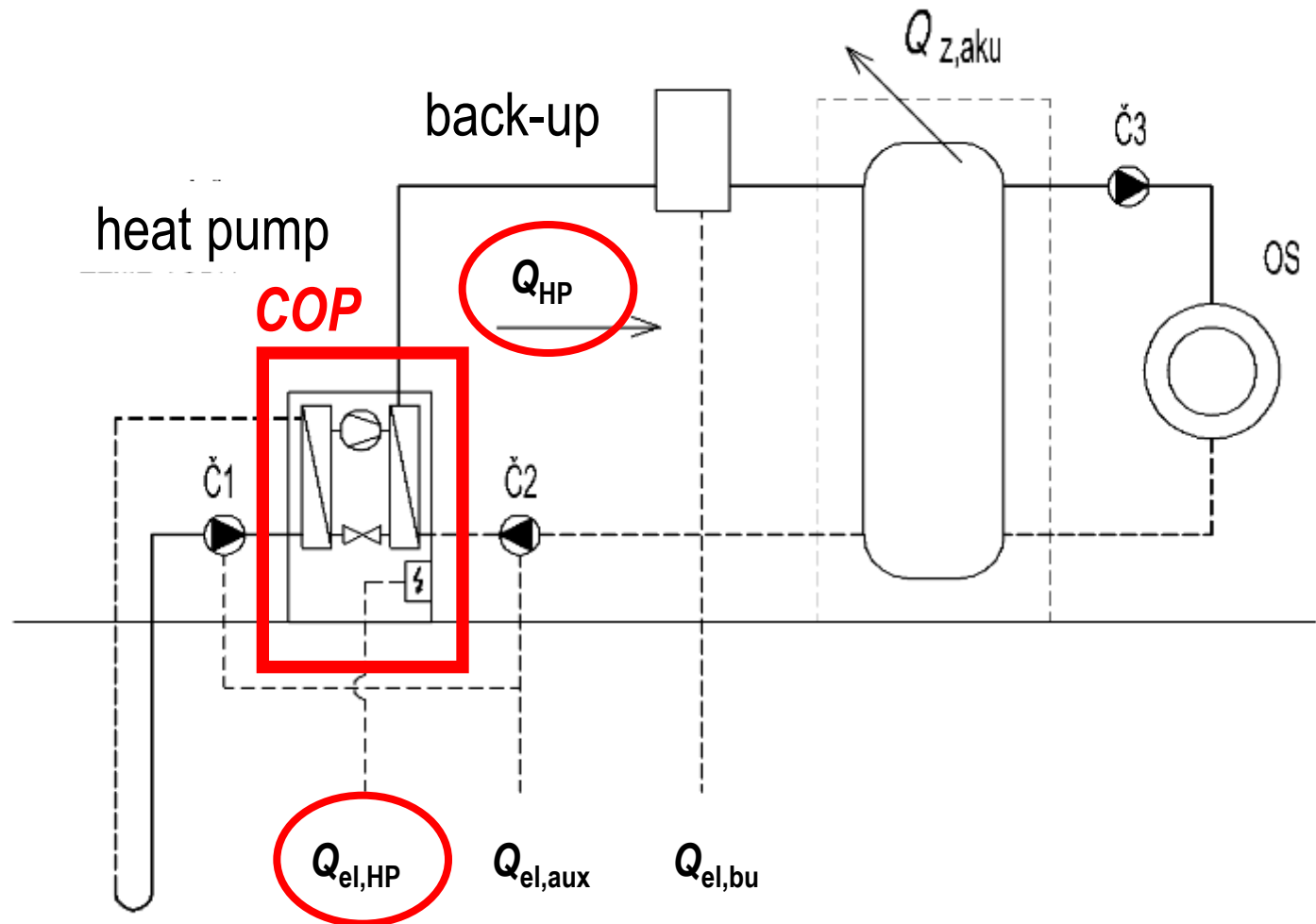


# performance factor

hot water

space heating

$$COP = \frac{Q_{HP}}{Q_{el,HP}}$$



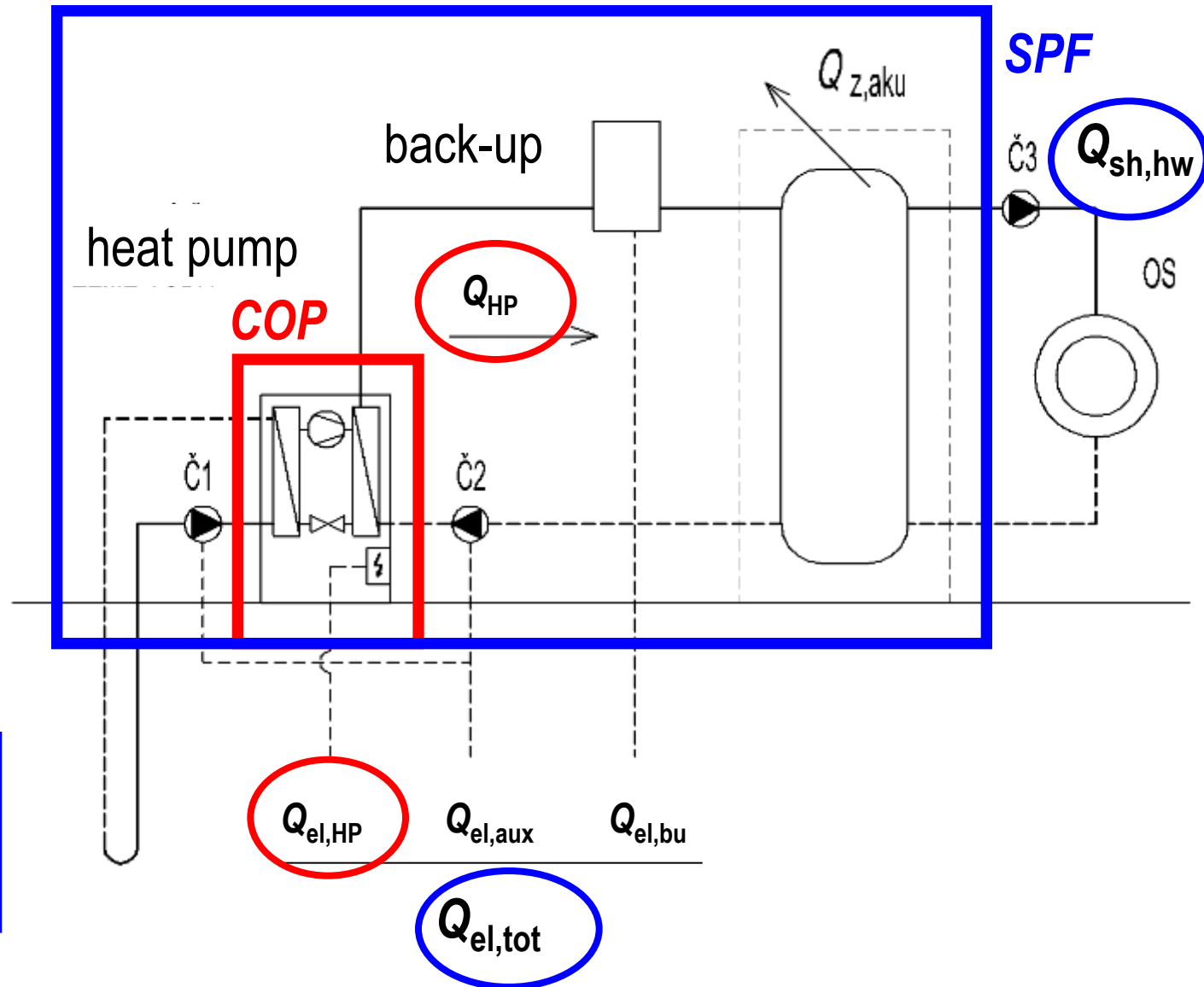


# Seasonal performance factor

hot water  
space heating

$$COP = \frac{Q_{HP}}{Q_{el,HP}}$$

$$SPF = \frac{Q_{sh,hw}}{Q_{el,tot}}$$





# RES directive, minimum SPF

---

- **heat pumps consume electric energy**
  - produced mainly from fossil fuels (primary non-renewable energy source)

$$SPF > 1,15 \frac{1}{\eta_e}$$

$\eta_e$  electricity production efficiency  
european average 45.5 %

***SPF > 2.5***

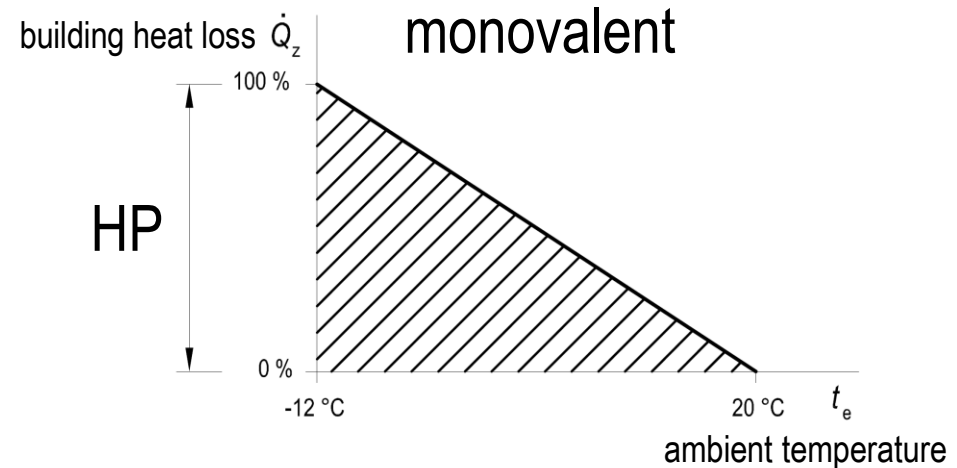
**if  $SPF < 2.5$  ... better to use fossil fuels directly by combustion**



# Operation modes

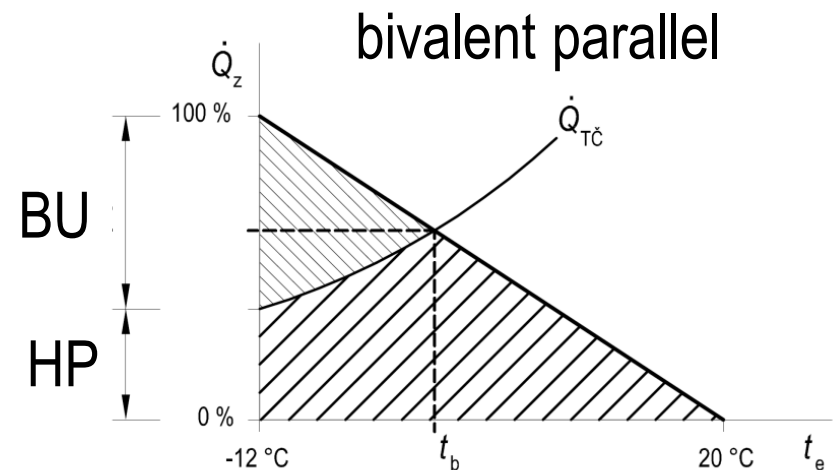
- **monovalent operation**

- only heating device



- **parallel bivalent operation**

- under bivalent temperature (**balance point**) back-up heater is switched-on
  - low temperature systems



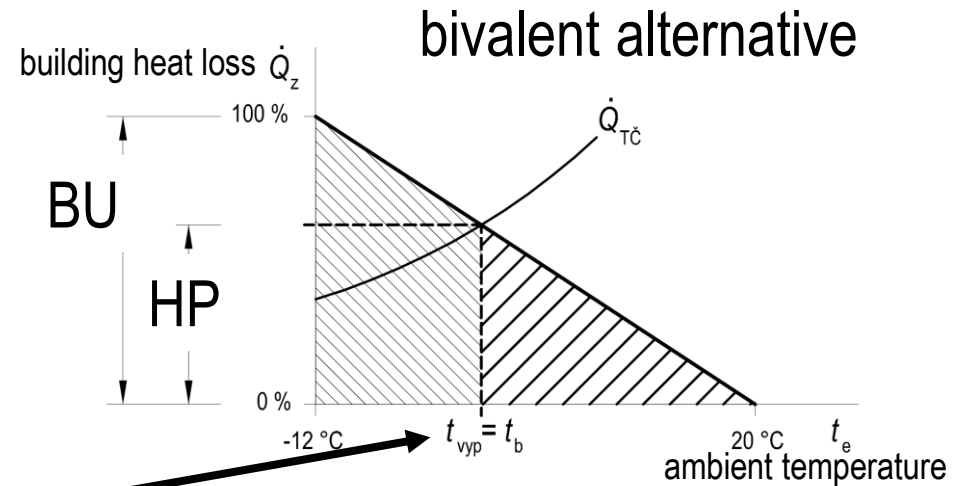
Heat production/loss in function of the outdoor (ambient) temperature



# Operation modes

- **alternatively bivalent operation**

- under bivalent temperature back-up replaced heat pump. for high temperature heating systems



**the balance point** = temperature under which the back-up heater is required



# Operation modes

---

- **monoenergetic operation**
  - e.g. bivalent operation of electric heat pump with electroboiler (integrated in one device)
  
- **balance point**
  - according to heat output (dimensioning)
  - acc. heating water temperature
    - sufficient heat output from heat pump
    - high temperatures of heating water needed, which couldnt be supplied by heat pump, esp. in extreme winter



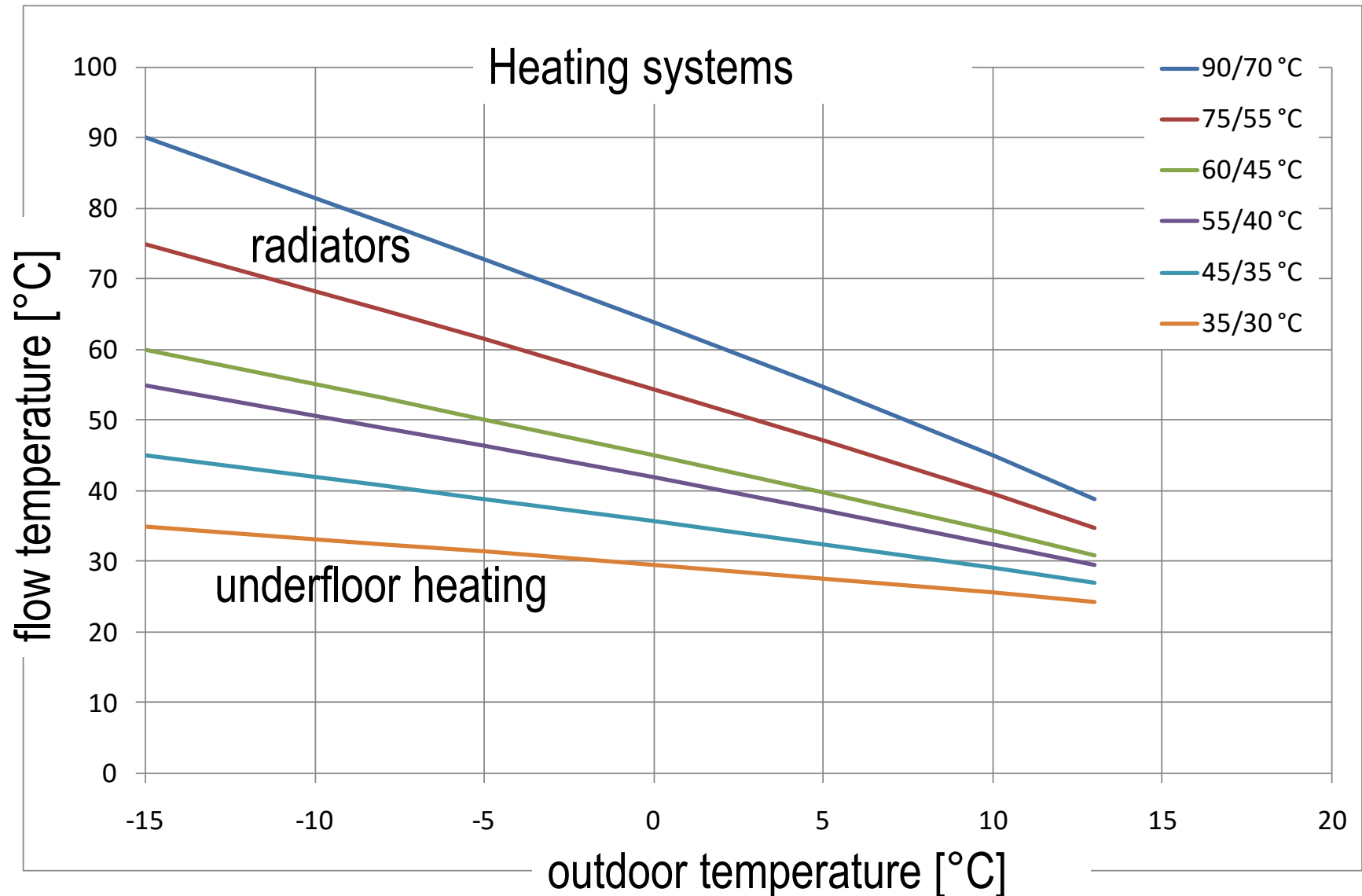
# Heat pump sizing

---

- **determination of heat pump type**
  - available heat source
  
- **determination of (condenser) heat output  $Q_k$  (for space heating)**
  - building heat loss
  - heat output for hot water
  - design flow temperature
  - design source temperature

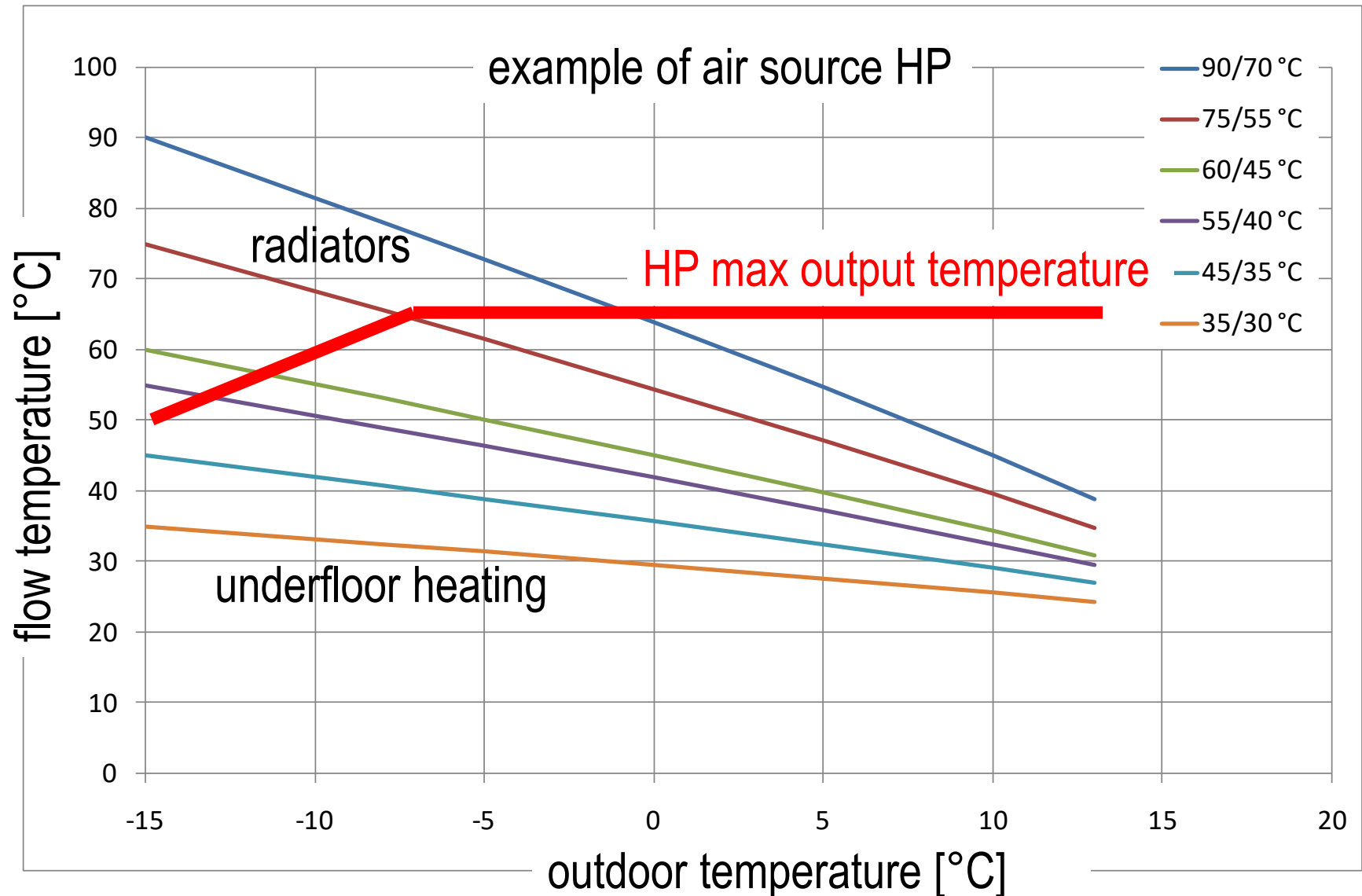


# Balance point according to temperature



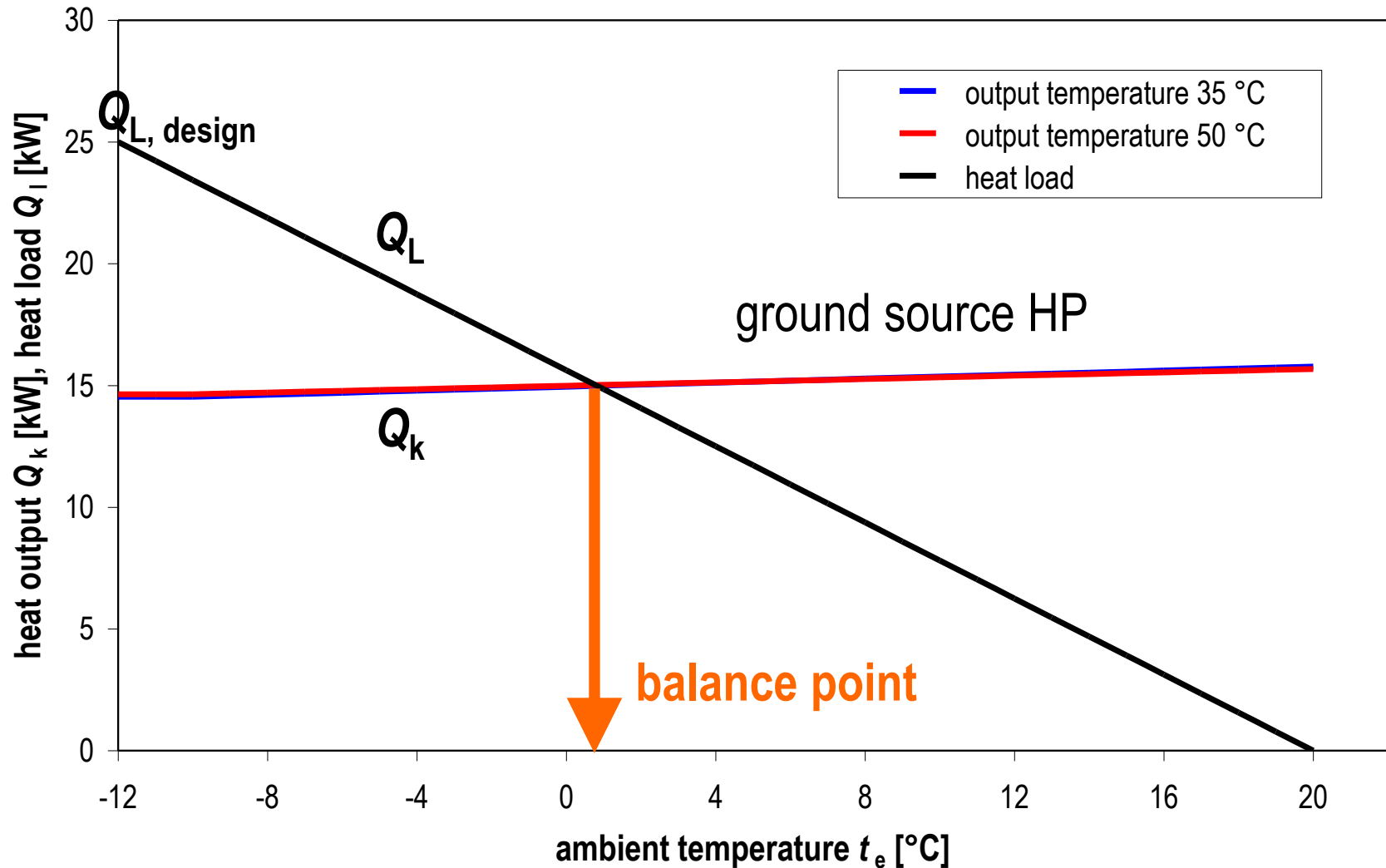


# Balance point according to temperature



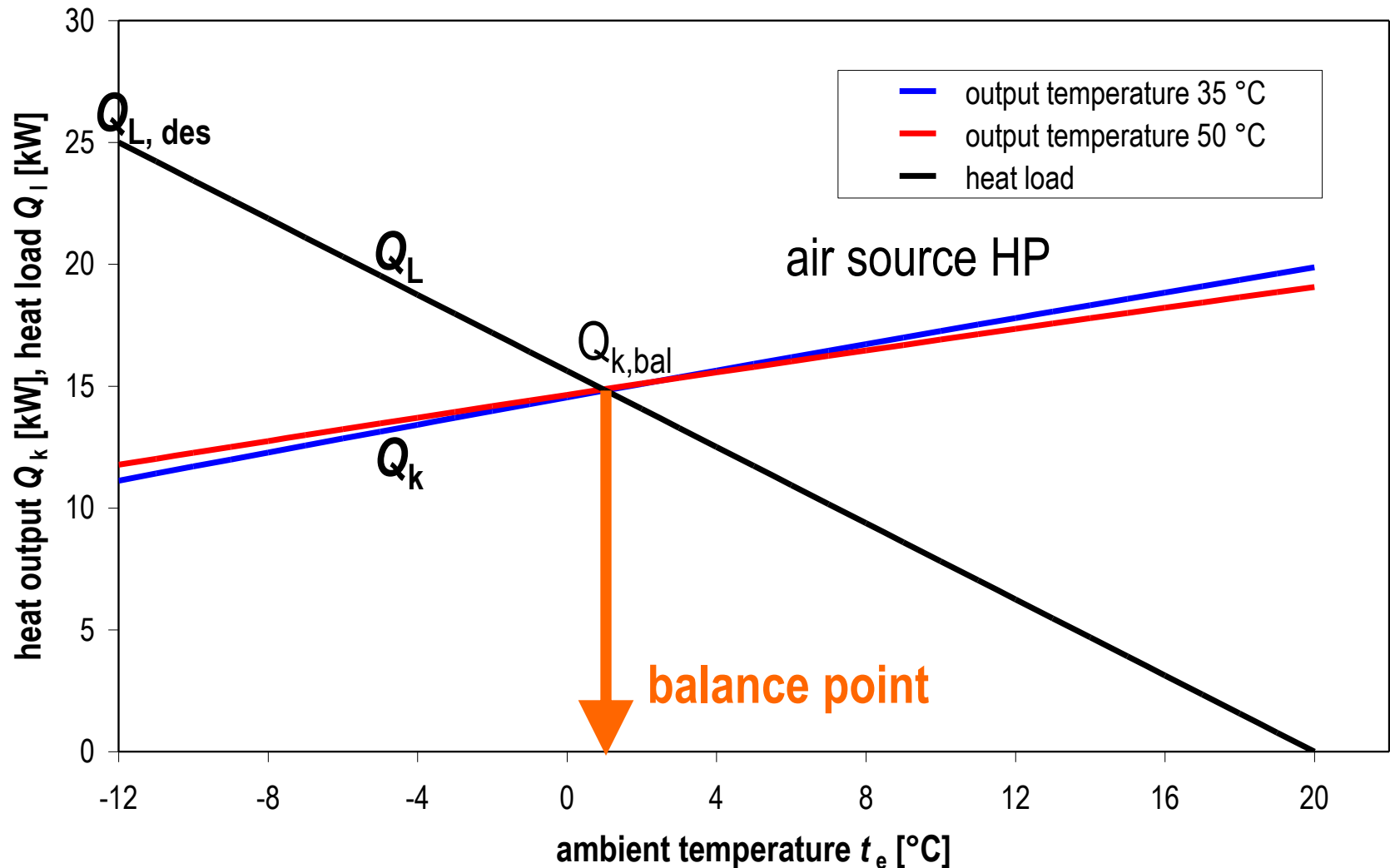


# Balance point according to heat output





# Balance point according to heat output





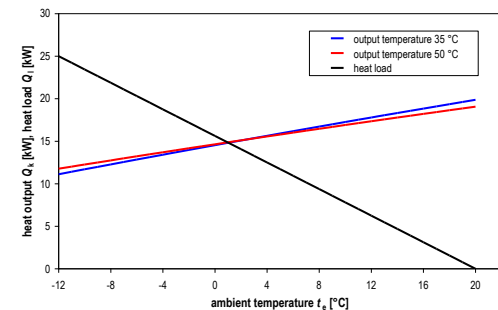
# Balance point determination

- design heat load  $Q_{L,des}$ 
  - calculation according to EN 12 831 for design external temperature (e.g. -12 °C, -15 °C, -18 °C in CZ)
- heat output at balance point  $Q_k = Q_L$ 
  - e.g. from desired fraction 60 to 100 %

$$\frac{Q_{L,des}}{(t_i - t_{e,des})} = \frac{Q_L}{(t_i - t_e)} = \frac{Q_{k,bal}}{(t_i - t_{e,bal})}$$

$$t_{e,bal} = t_i - \frac{Q_{k,bal}}{Q_{L,des}} (t_i - t_{e,des})$$

$$Q_{k,bal} / Q_{L,des}$$



if  $Q_k = \text{konst}$

if  $Q_k \neq \text{konst}$  - graph



# Example

family house

design load 15 kW for  
design temperature  $-15^{\circ}\text{C}$

heat pump (SE WPL18)

heating system  $50/40^{\circ}\text{C}$

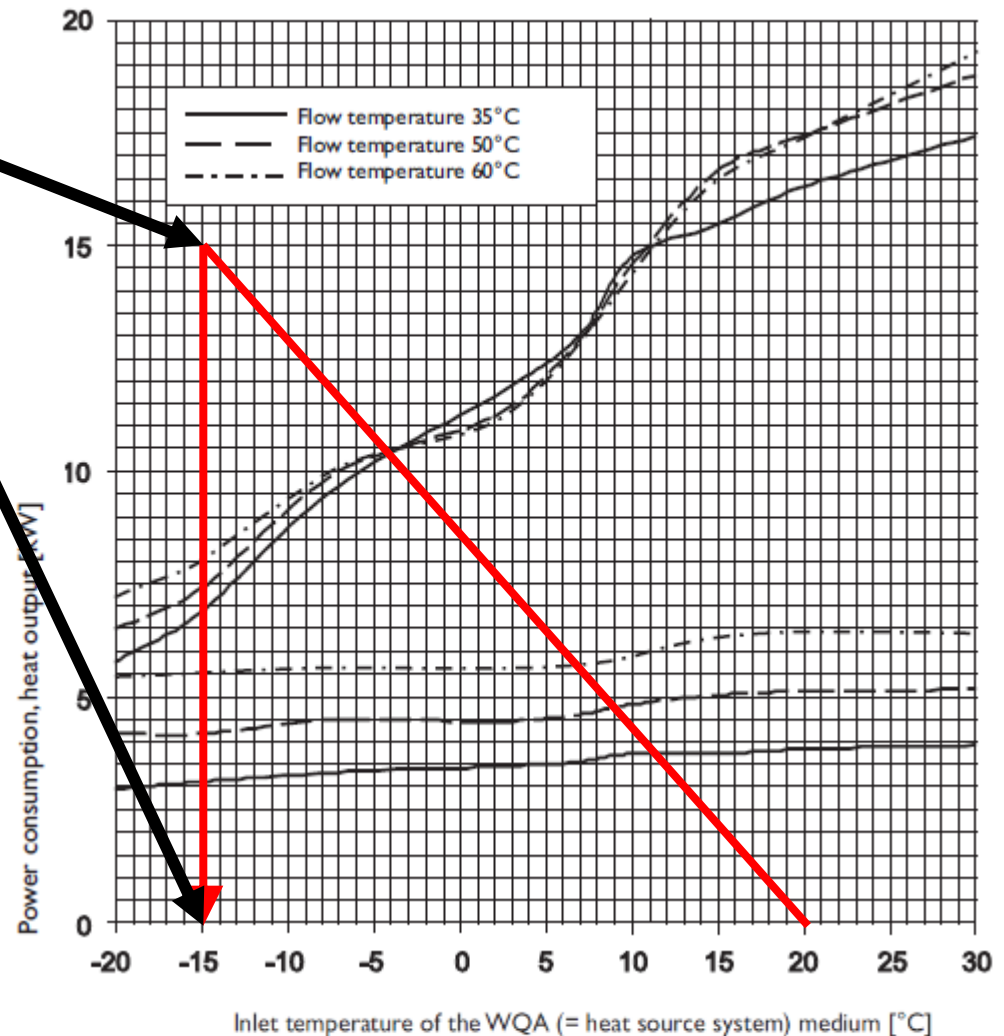
determine the balance point

balance point heat output

balance point power input

balance point COP

Output diagram for heat pump WPL 18





# Example

family house

design load 15 kW for

design temperature -15 °C

heat pump (SE WPL18)

heating system 50/40 °C

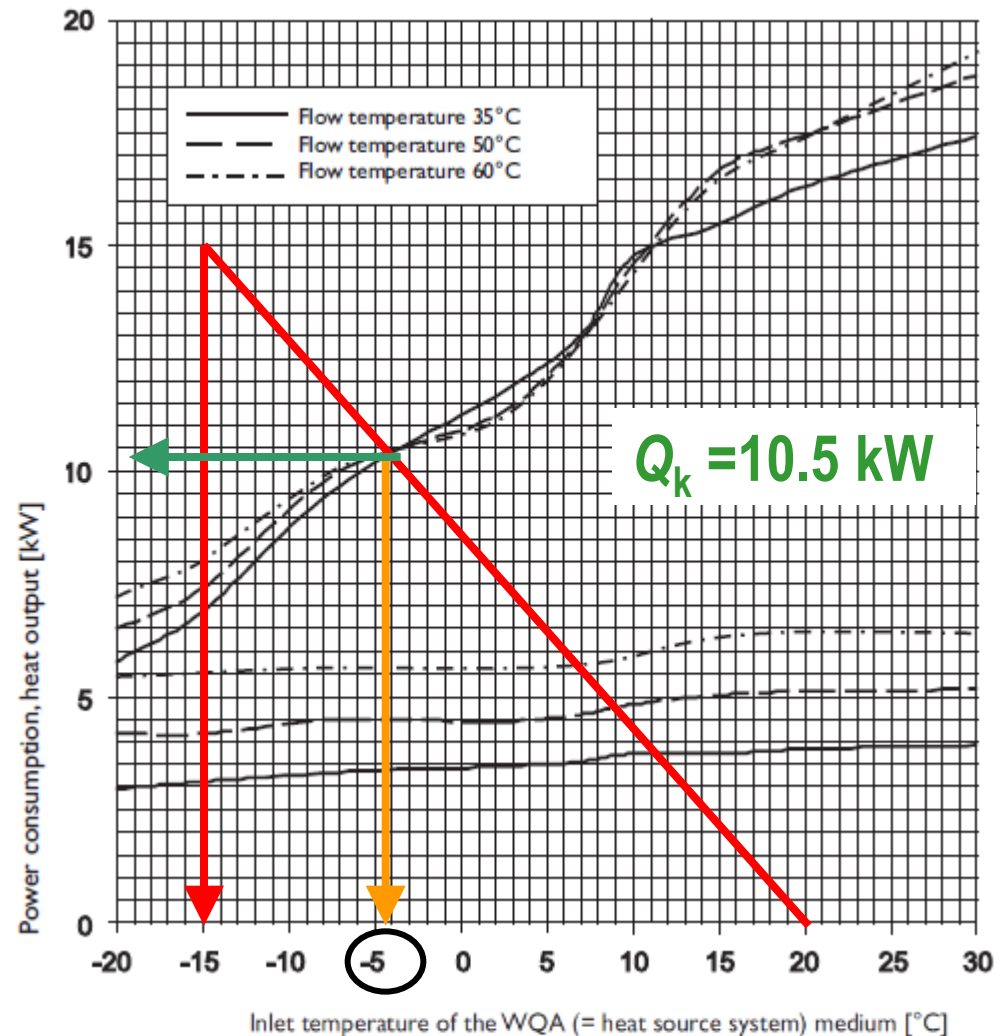
determine the balance point

balance point heat output

balance point power input

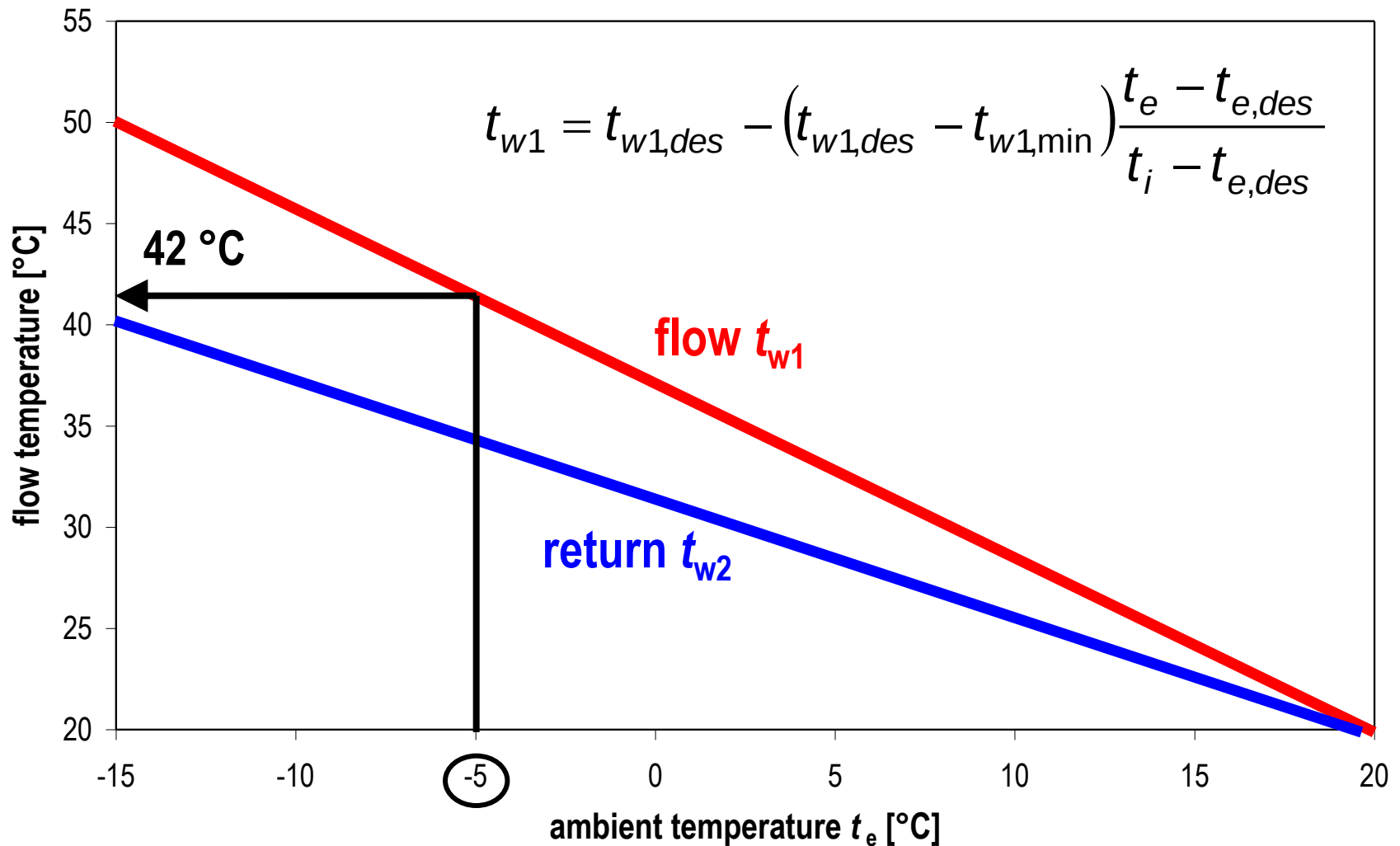
balance point COP

Output diagram for heat pump WPL 18





# Flow temperature





# Power input at balance point

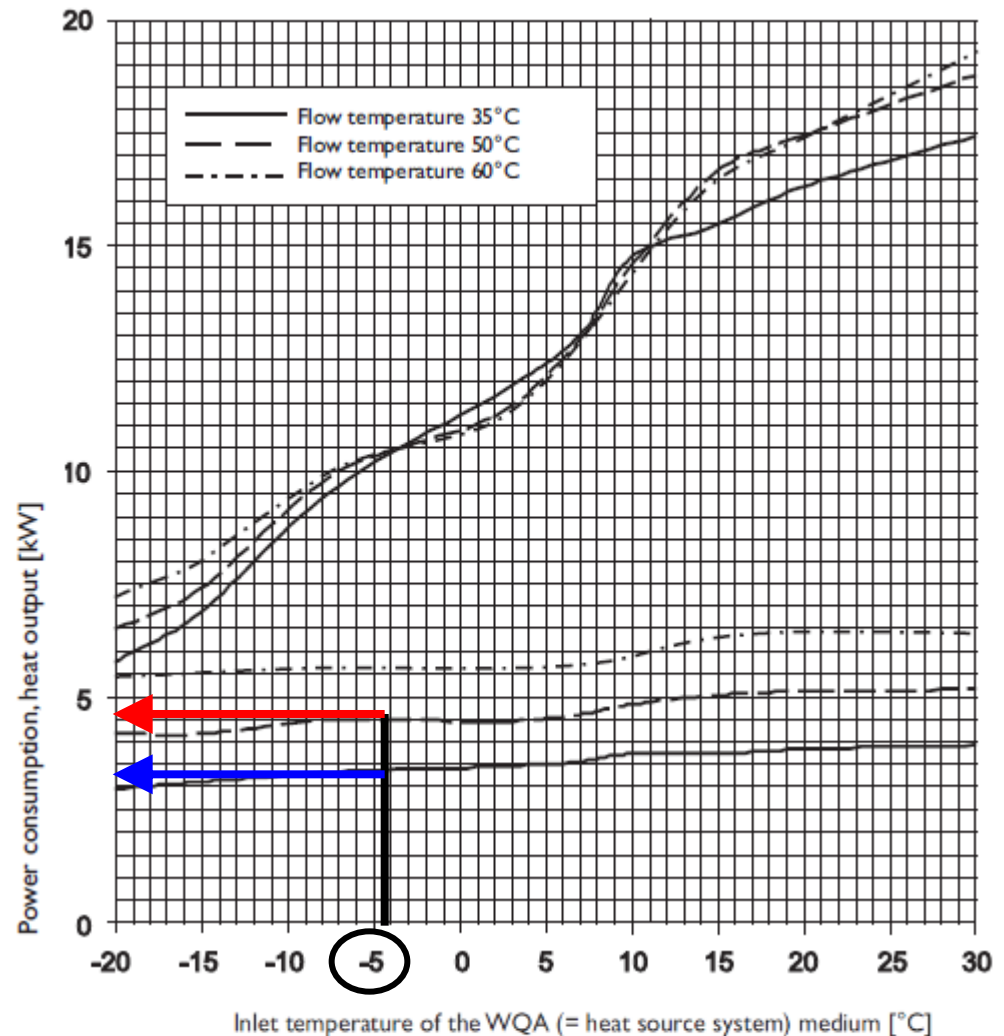
for balance power point

$$t_{v1} = -5\text{ °C}, t_{k2} = 42\text{ °C}$$

$$P_{35} = 3.3\text{ kW}$$

$$P_{50} = 4.5\text{ kW}$$

Output diagram for heat pump WPL 18





# Power input at balance point

for balance power point

$$t_{v1} = -5 \text{ }^{\circ}\text{C}, t_{k2} = 42 \text{ }^{\circ}\text{C}$$

$$P_{35} = 3.3 \text{ kW}$$

$$P_{50} = 4.5 \text{ kW}$$

interpolation

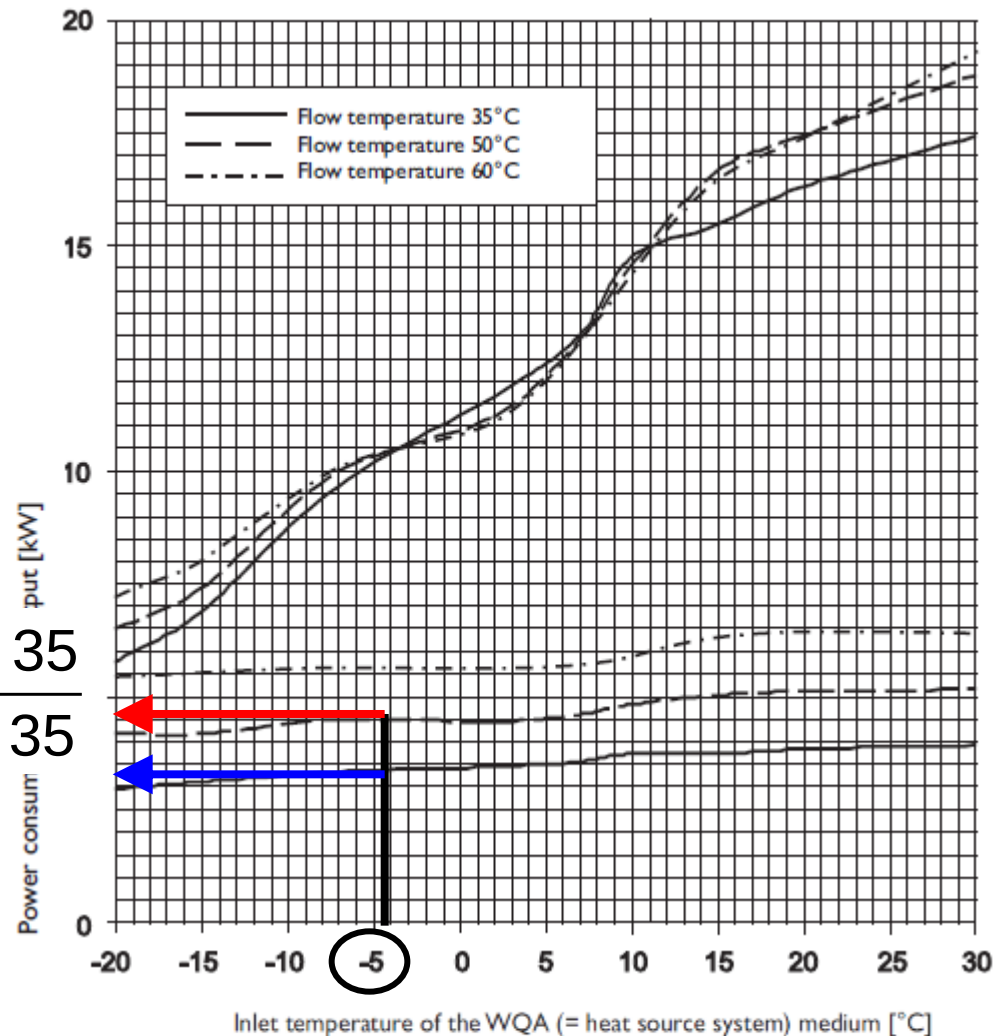
$$\frac{P_{tw1} - P_{35}}{t_{w1} - 35} = \frac{P_{50} - P_{35}}{50 - 35}$$

$$P_{tw1} = P_{35} + (P_{50} - P_{35}) \frac{t_{w1} - 35}{50 - 35}$$

balance power point

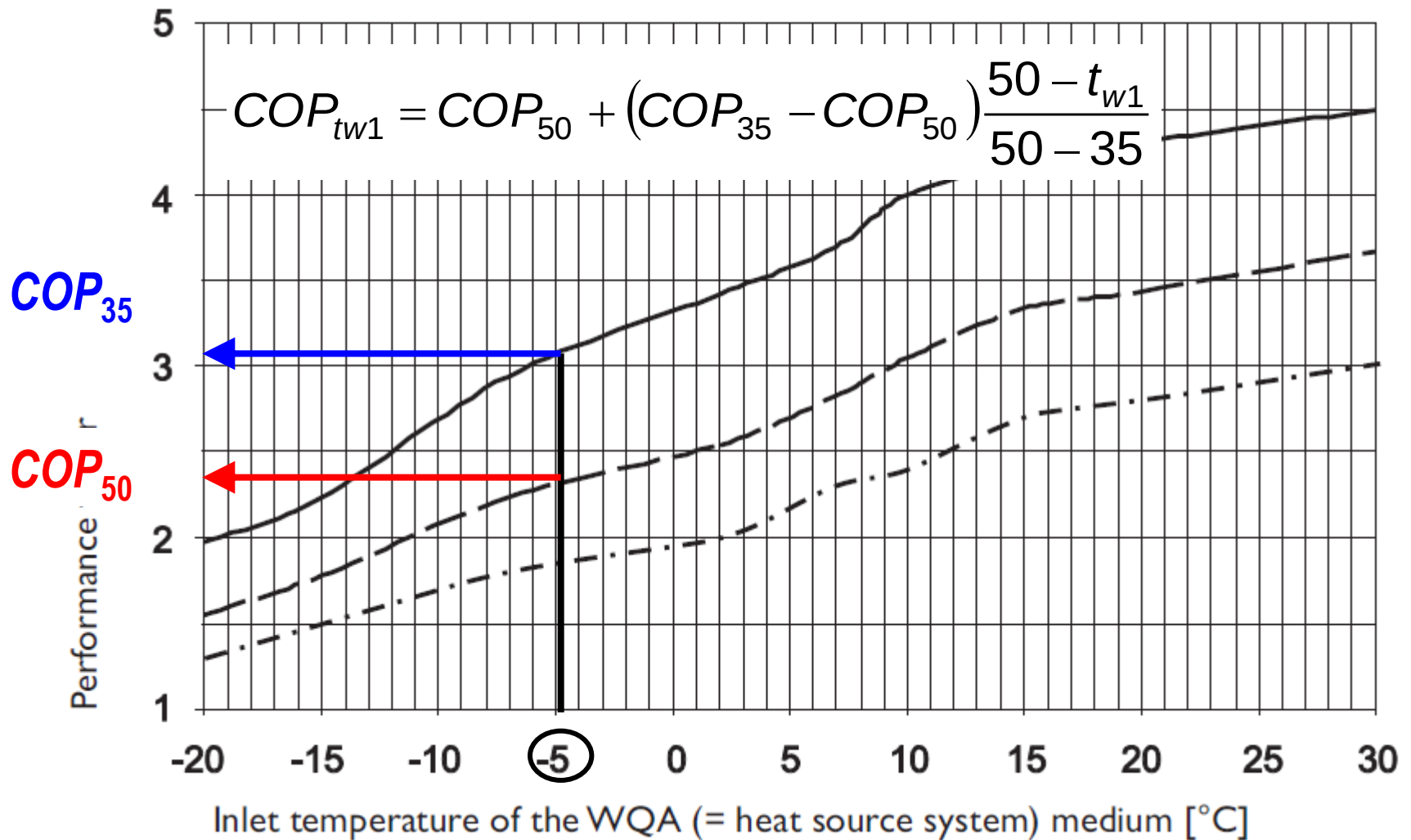
$$P_{42} = 3.9 \text{ kW}$$

Output diagram for heat pump WPL 18





# Example





# Example

family house

design load 15 kW for

design temperature -15 °C

heat pump (SE WPL18)

heating system 50/40 °C

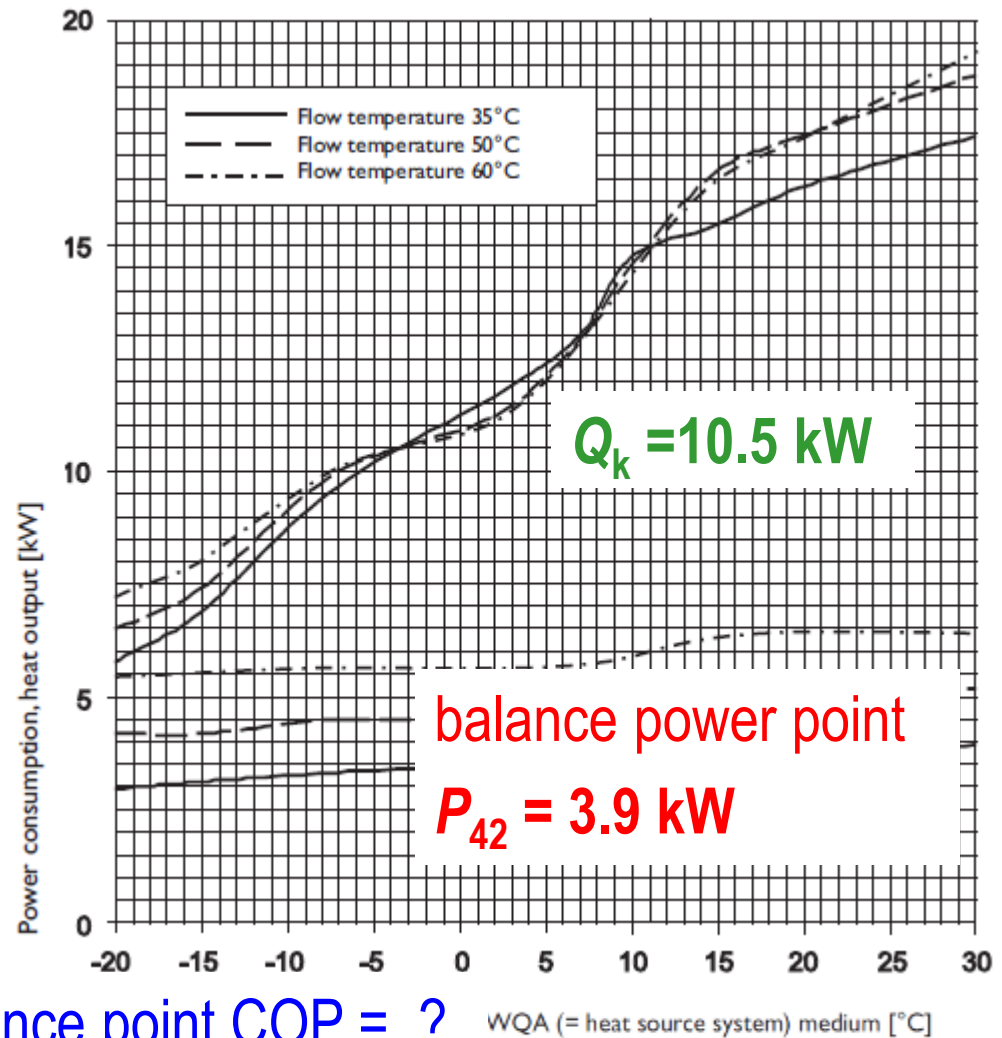
determine the balance point

balance point heat output

balance point power input

balance point COP

Output diagram for heat pump WPL 18



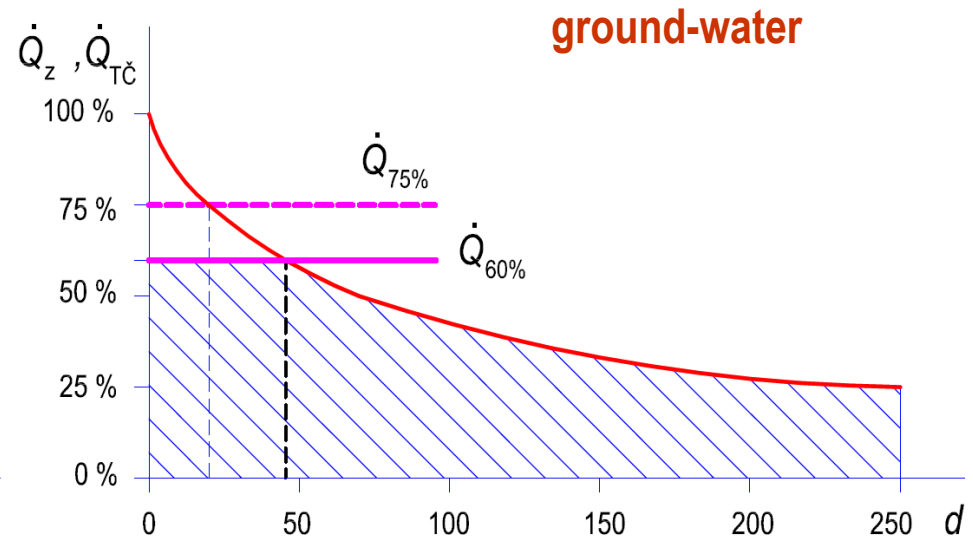
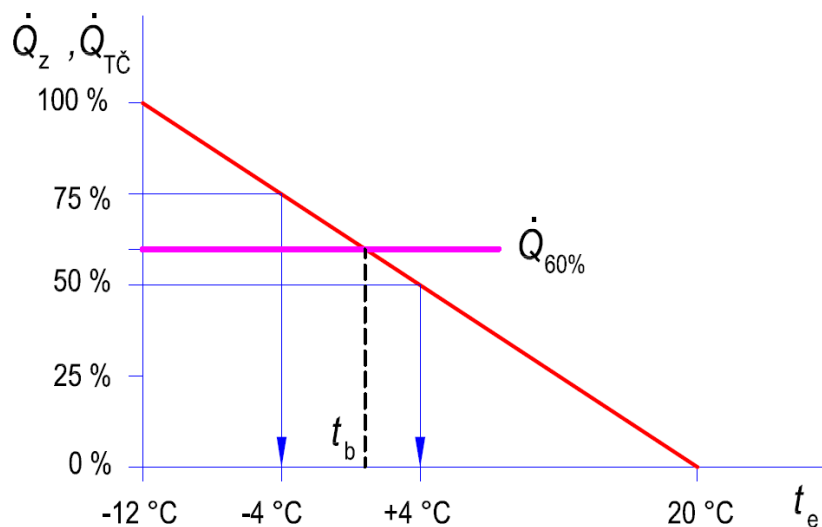
balance point COP = ?





# Heat pump sizing (ground source)

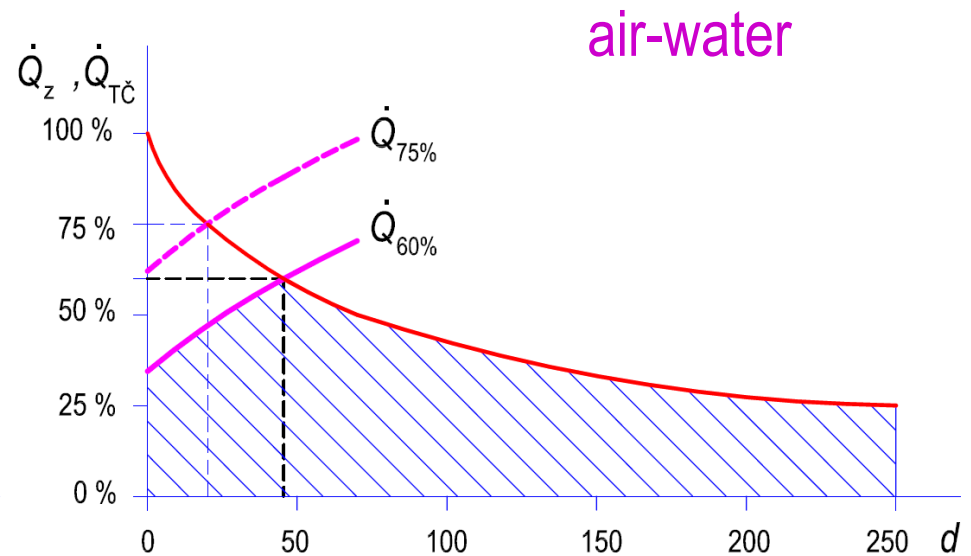
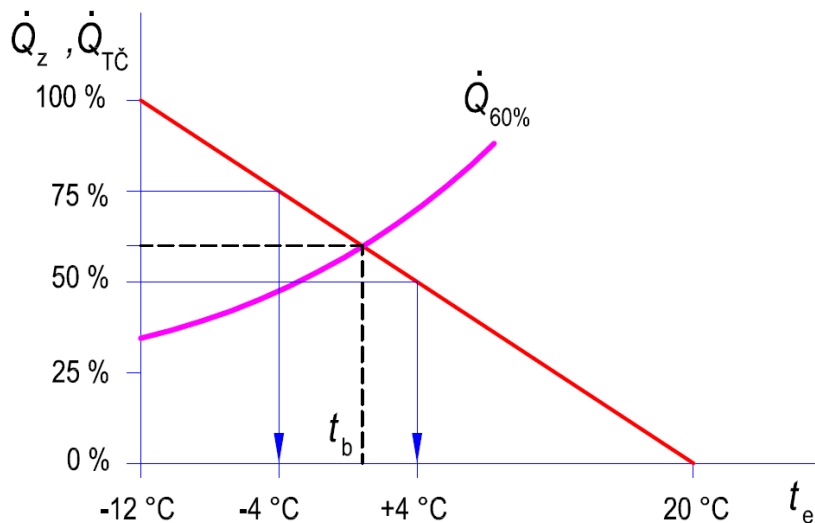
- dimensioning heat output (independent on ambient conditions)
  - 50 % heat loss - coverage 85 % heat demand
  - 60 % heat loss – coverage 93 % heat demand
  - 70 % heat loss - coverage 97 % heat demand





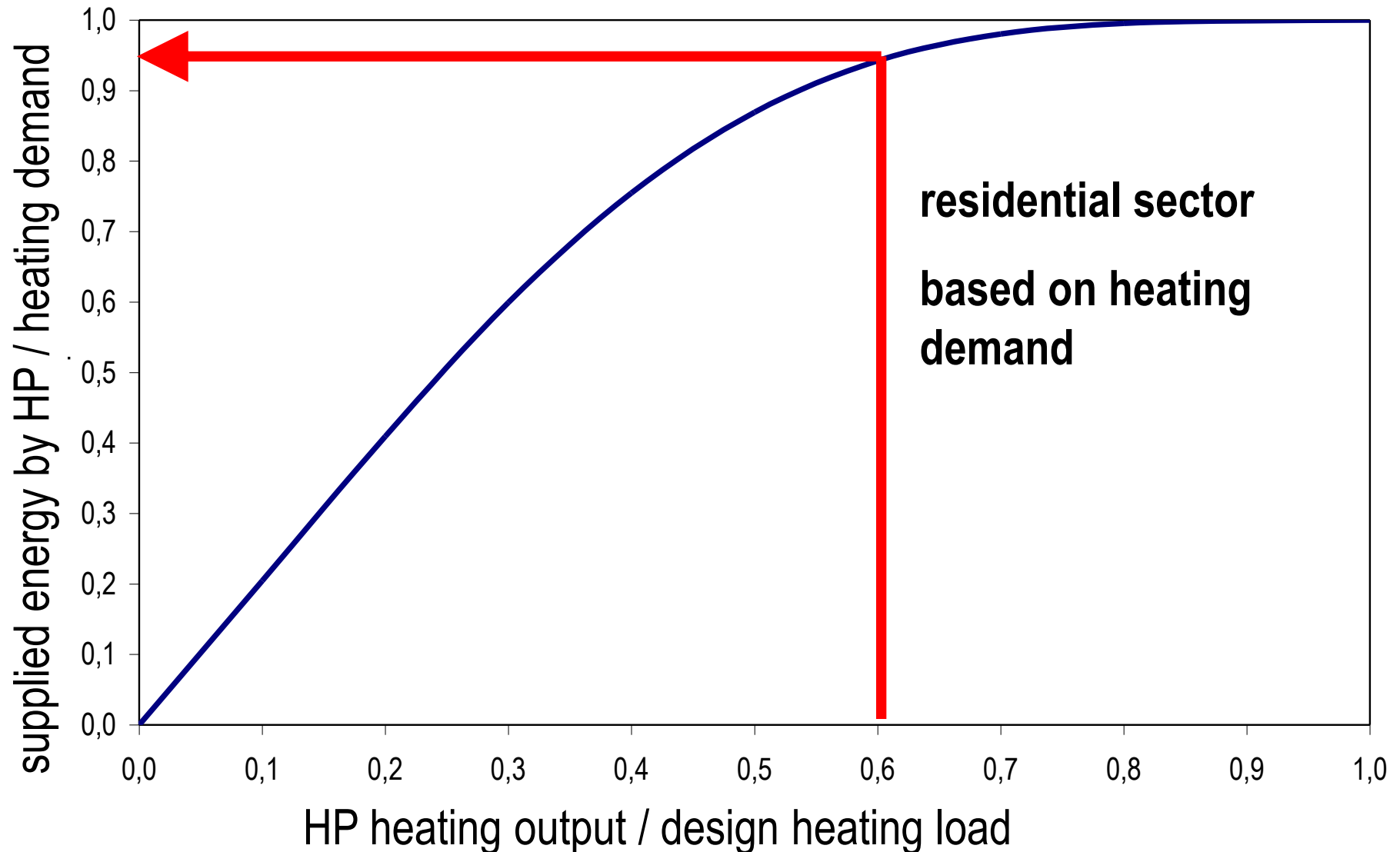
# Heat pump sizing (air source)

- dimensioning heat output (dependent on ambient conditions)
  - 50 % heat loss - coverage 75 % heat demand
  - 60 % heat loss – coverage 85 % heat demand
  - 70 % heat loss - coverage 92 % heat demand





# Heat pump sizing – coverage of demand





# Heating capacity control

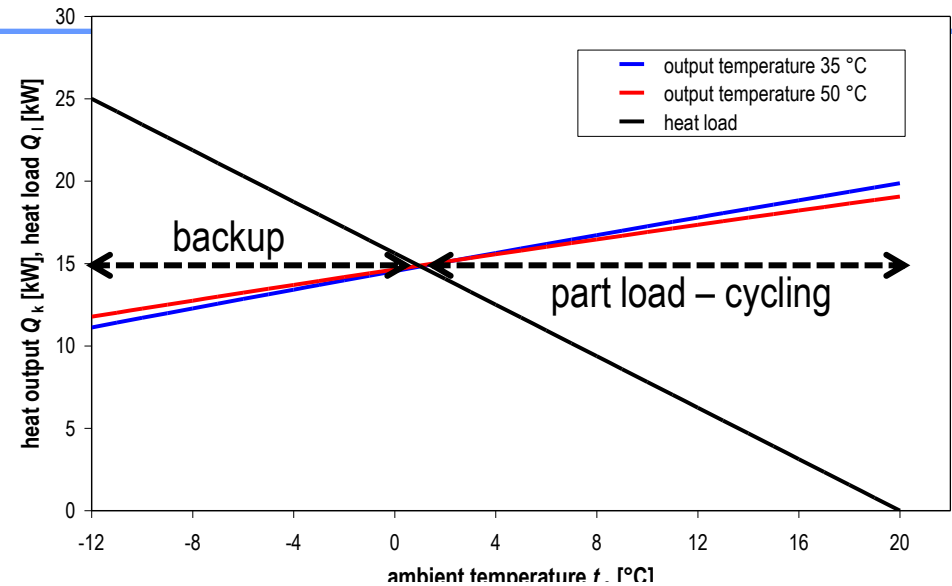
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- **start-stop regime heat pumps**
  - cycling = reduction of durability compressor
  - elimination cycling
    - undersizing (possible?)
    - heat storage – sizing of store for minimum operation time of heat pump
- **heat pump with heating capacity control**
  - **compressor speed control**
  - possibility for monovalent operation

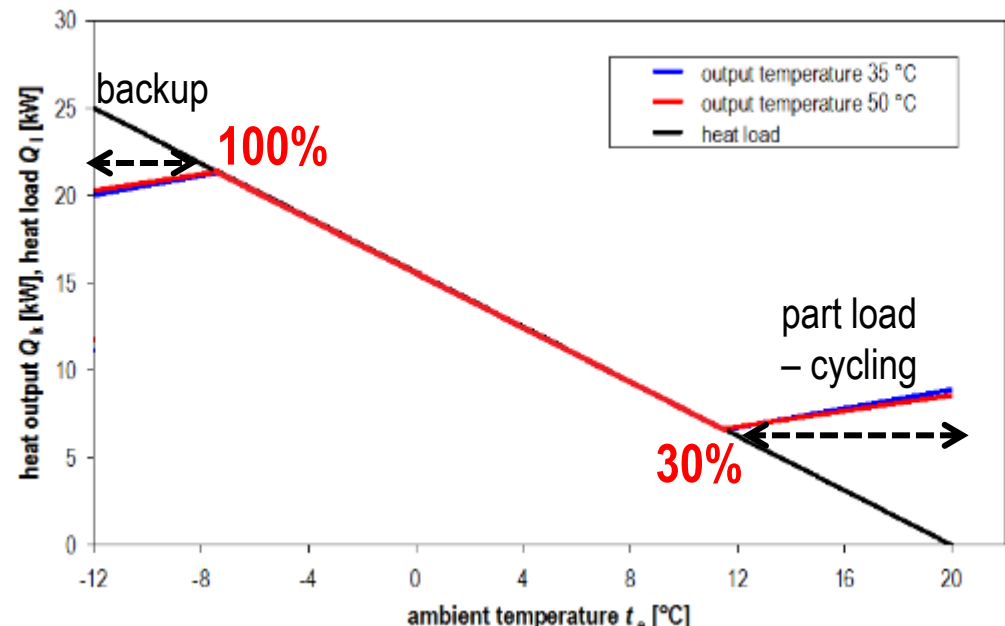


# Heating capacity control

Without capacity control



With capacity control





# Heat storage for heat pump

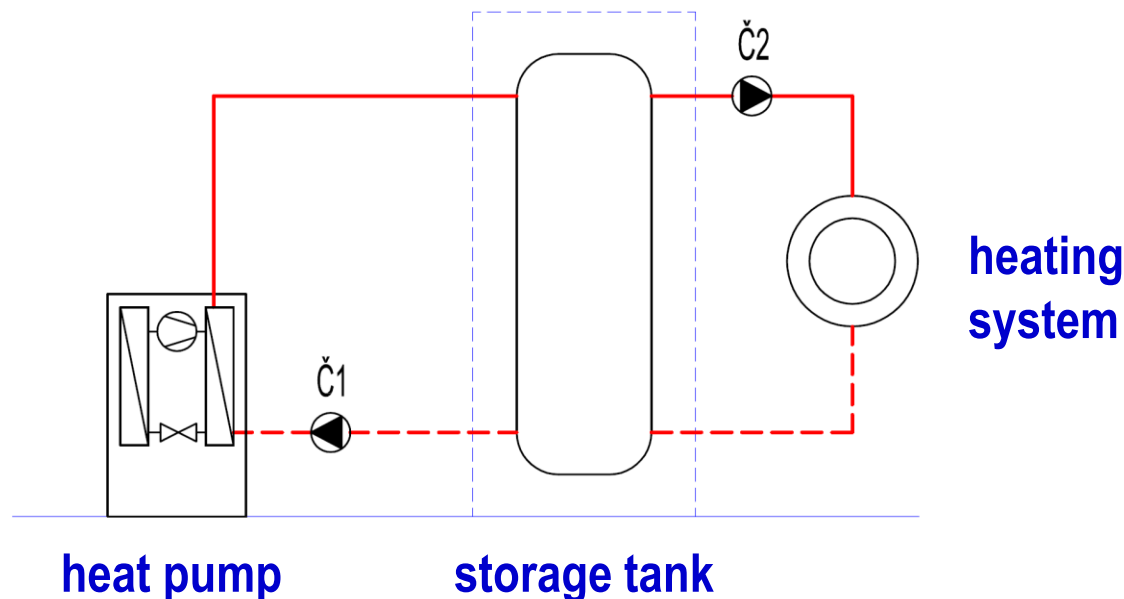
---

- **oversized storage** for most of operation time
  - balancing heat output and heat load
- **reduction of frequency compressor on/off** (1 x 10 min)
  - longer durability of compressor
- **heat source for outdoor units** (air-water)
  - antifreeze protection



# Heat storage for heat pump

- **hydraulic decoupling of heat pump** from heating circuit
  - hydraulic shunt
  - heating systems can't influence HP circuit
  - providing required (higher) flowrates at condenser





# Sizing of storage tank

---

- **balance** to reduce on/off frequency
  - minimum operation time period  $\Delta\tau$
  - increase of temperature  $\Delta t$  in store during operation of heat pump
- heat stored during operation of heat pump

$$Q_{\text{stored}} = \dot{Q}_{HP} \cdot \Delta\tau = V \cdot \rho \cdot c \cdot \Delta t$$



# Sizing of storage tank

---

specific volume

$$\frac{V}{\dot{Q}_{HP}} = \frac{1000 \cdot \Delta \tau}{\rho \cdot c \cdot \Delta t}$$

$\Delta \tau$  [s]      minimum operation time:      15 min

$\Delta t$  [K]      increase of store temperature:      3 - 5 K

usually      **15 to 30 l/kW**

thermal capacity (momentum) of heating system results in lower volume requirement



## Example

- **calculate** heat storage volume for balance point heat output **10.5 kW**
  - minimum operation time period  $\Delta\tau = 15$  min
  - increase of temperature  $\Delta t$  in store 3 K

$$V = \dot{Q}_{HP} \frac{1000 \cdot \Delta\tau}{\rho \cdot c \cdot \Delta t}$$

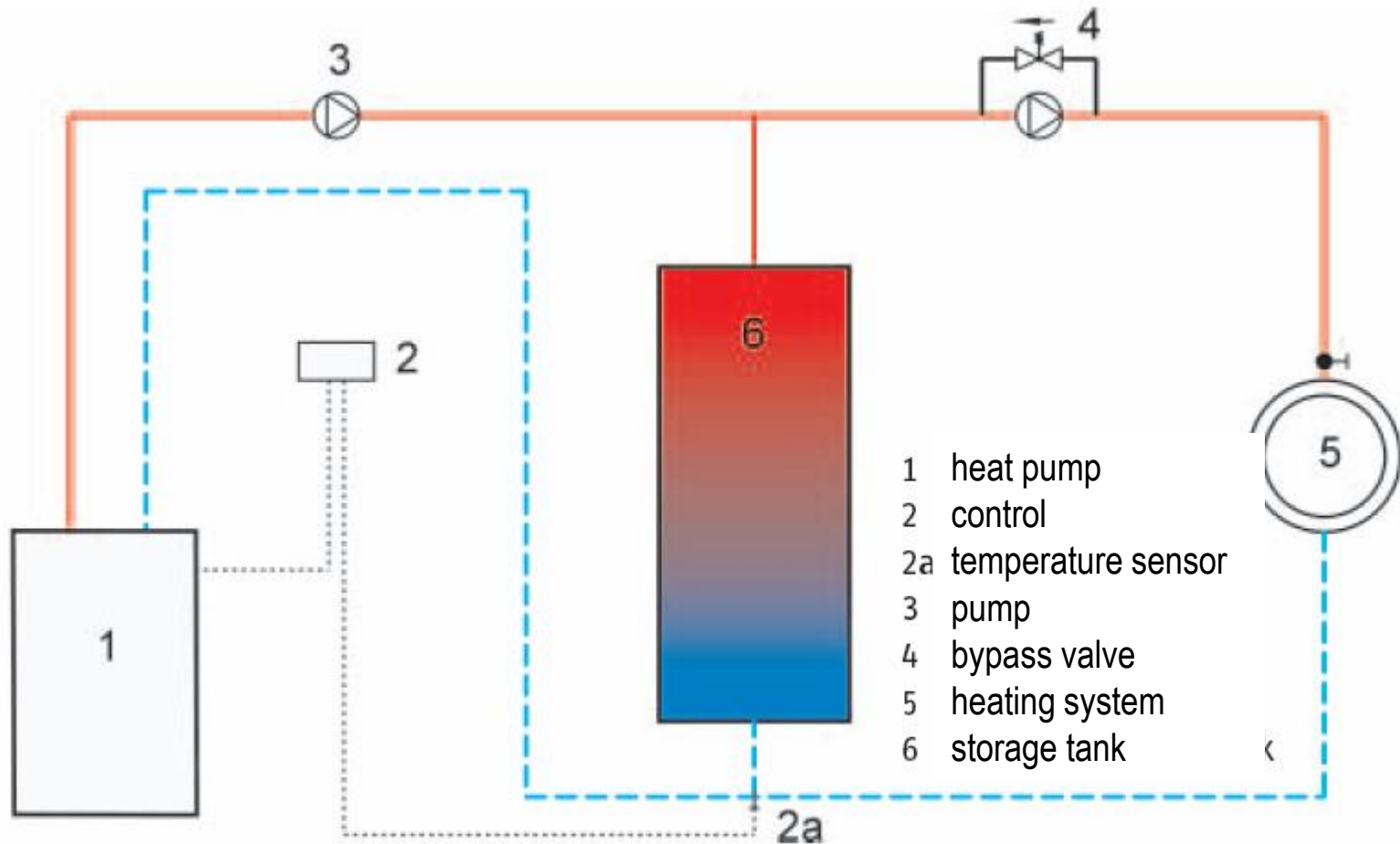
$$V \text{ [m}^3\text{]} = \dot{Q}_{HP} \text{ [kW]} \cdot \frac{1000 \cdot 15 \cdot 60}{998 \cdot 4187 \cdot 3}$$

$$V = 754 \text{ l}$$



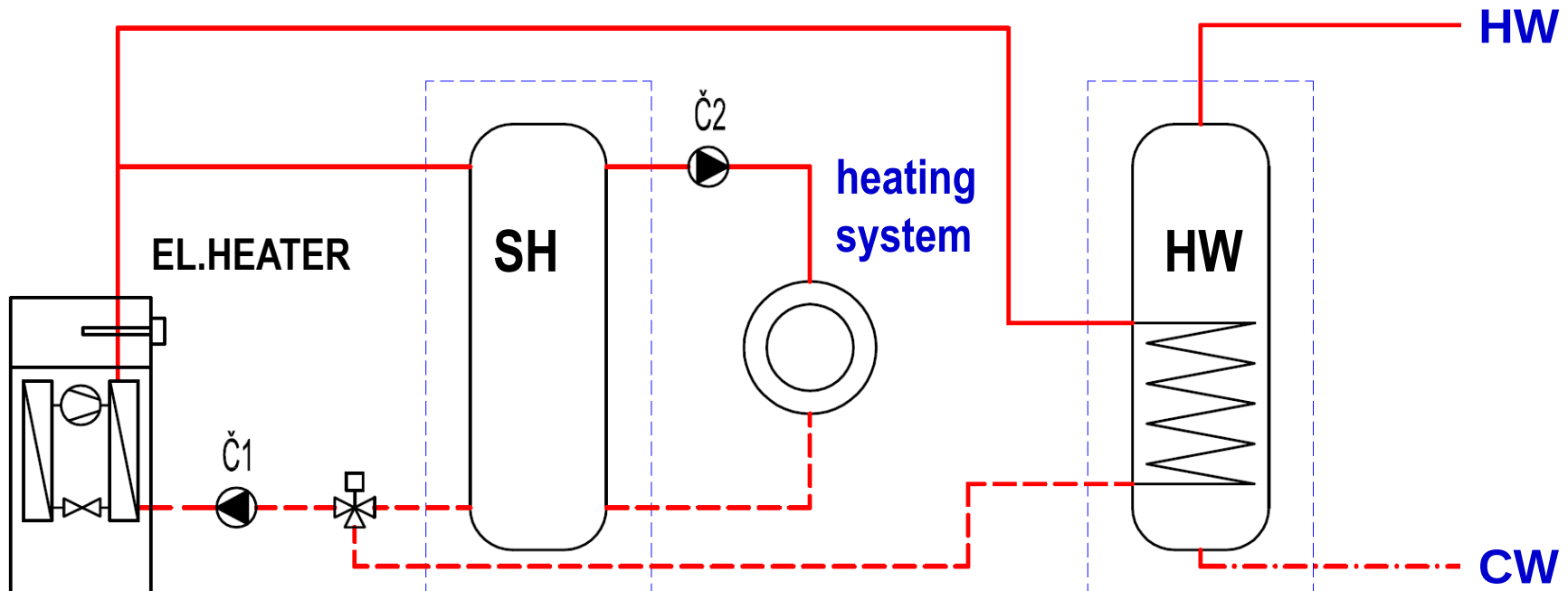
# Integration of store

electronically  
controlled pump



# Hydraulics

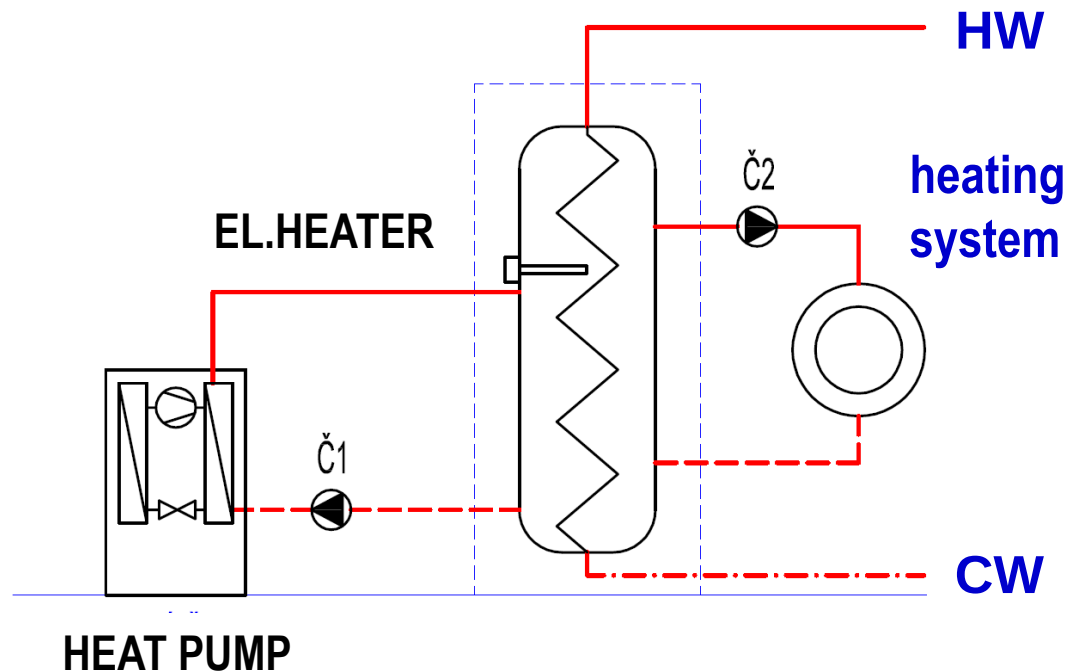
- **with two stores**
  - heating water store, hot water store
  - bivalent (back-up) heater inside heat pump





# Hydraulics

- **with central store**
  - hot water heated in heat exchanger immersed in heating water store volume
  - bivalent (back-up) heater: immersed in store





# Guides for design

---

- **low temperature heating  $< 45\text{ }^{\circ}\text{C}$** 
  - floor heating, wall heating
  - radiators with larger surface
  
- **pool water heating**
  
- **hot water**
  - low temperature  $45\text{ }^{\circ}\text{C}$
  - air-water: advantage in summer, high ambient temperatures
  - brine-water: reduction of borehole regeneration (!)